

# Vibration Energy Harvesting

## Basics

Paul D. Mitcheson

*NIPS workshop 2014*

# Overview

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- Motivation
- EH Overview
- Transducers Overview
- Fundamental Analysis and Optimisation
- System Modelling and Simulation Approaches
- Transducer and Power interface Requirements
- Bringing it together - piezoelectric Example
- Conclusions

*These systems fundamentally have strong coupling between electrical and mechanical parts – hence even a basic overview must consider the electronic load*

## Imperial - history

**1851–1890** Constituent Colleges formed,  
Prince Albert and the Great Exhibition

**1907** Imperial College founded by merger of:

- City and Guilds College
- Royal College of Science
- Royal School of Mines

**1988-2000** Mergers with:

- St Mary's Hospital Medical School
- National Heart & Lung Institute
- Charing Cross/Westminster and  
Royal Postgraduate Medical Schools
- Kennedy Institute

**2007** Left the University of London to become an  
independent university



## Past achievements

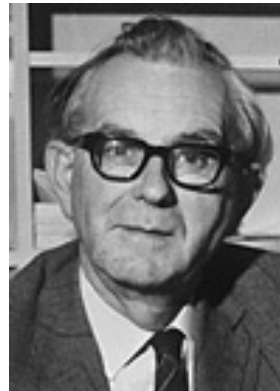
14 Nobel Prize Winners associated with Imperial College



Alexander  
Fleming:  
**Penicillin**



Denis  
Gabor:  
**Holography**



Rodney  
Porter:  
**Structure of  
Antibodies**



Andrew  
Huxley:  
**Nerve  
Impulses**



Abdus  
Salam:  
**Theoretical  
Physics**

# Imperial College London



Imperial College  
London



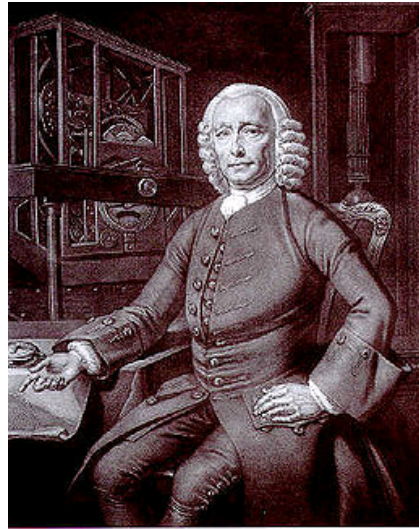
Electrical and Electronic Engineering Dept  
11 story building – spot the cleanrooms...

## But I'm not from London Originally

- When you ask an Italian born in Rome what nationality he is, he will say he is Roman, not Italian.
- People from Yorkshire are similar...
- They are also suborn and talk with funny accents (I have been away for too long..)



# What is Yorkshire Famous for?

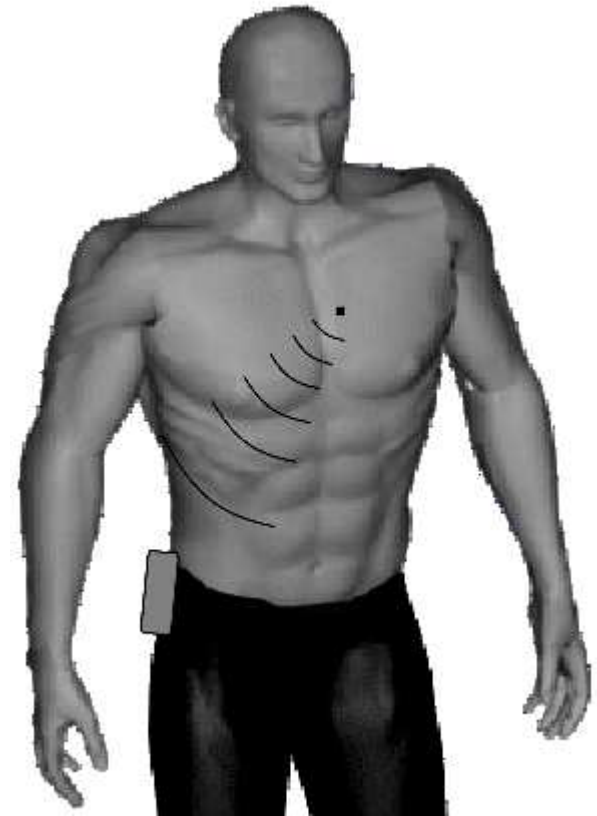


# **EH Overview**



## Energy Harvesting – Original Motivations

- Energy harvesting – replacement of finite power sources in portable devices by converting ambient energy into electricity through the use of an energy harvester
- The field started to emerge in about 2000
- Needs knowledge on devices, physics and electronics
- A very small number of people had looked at some type of harvesting from high frequency vibration
- My original motivation was to look at human body powering of implantable sensors



## Orders of Magnitude of Power

World electrical generation capacity	4 terawatts
Power station	1 gigawatt
House	10 kilowatts
Person, lightbulb	100 watts
Laptop, heart	10 watts
Cellphone power usage	1 watt
Wristwatch, sensor node	1 microwatt
Received cellphone signal	1 nanowatt

*Sensor nodes are receiving a lot of interest – but the constraints on volume and power are significant and we must push design to the limit*

## Sources for Harvesting

<b>Energy Source</b>	<b>Conversion Mechanism</b>
<b>Light</b> Ambient light, such as sunlight	<b>Solar Cells</b>
<b>Thermal</b> Temperature gradients	<b>Thermoelectric or Heat Engine</b>
<b>Magnetic and Electro-magnetic</b> Electro-magnetic waves	<b>Magnetic induction (induction loop)</b> <b>Antennas</b>
<b>Kinetic</b> Volume flow (liquids or gases) Movement and vibration	<b>Magnetic (induction)</b> <b>Piezoelectric</b> <b>Electrostatic</b>

## Why choose Kinetic/Vibration Devices?

Original reasons:

- Thermal gradients are small at small sizes on body
- In most places (unless close to a base station) RF energy is quite limited
- Solar is no good for implanting into the body
- So movement seemed a logical choice
- Open research with lots of interesting questions



*Seiko stopped production of the thermic watch but continue the kinetic device*

## Motion Energy Harvesting

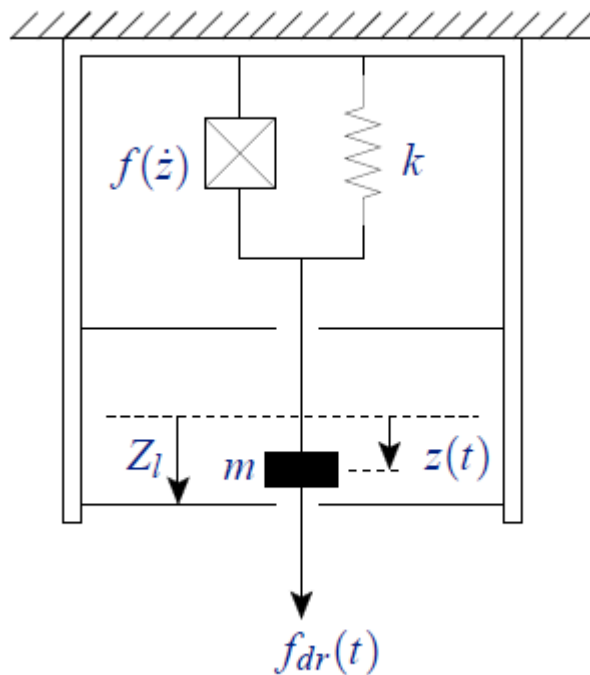
- Direct force devices (like most electrical generators)



- Inertial devices (most energy harvesters are of this form)

*This is a very important distinction...*

## Direct Force Device



- A force applied to a transducer
- Energy generated is force-distance integral
- Transducer could be piezoelectric, electrostatic or electromagnetic
- System dynamics probably dominated by driving force,  $f_{dr}(t)$
- If driving force is large enough, then make the damping as big as possible

## Direct Force Generators

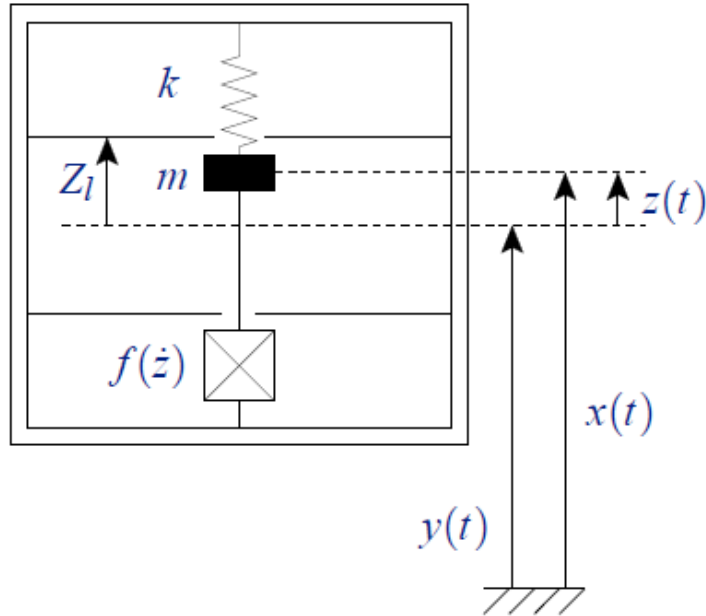


Heel strike generator: Paradiso et al,  
MIT

East Japan Railway Co.

- Energy harvesting ticket gates

# Inertial Generators



*We are going to spend a lot of time with this "mass in a box" model...*

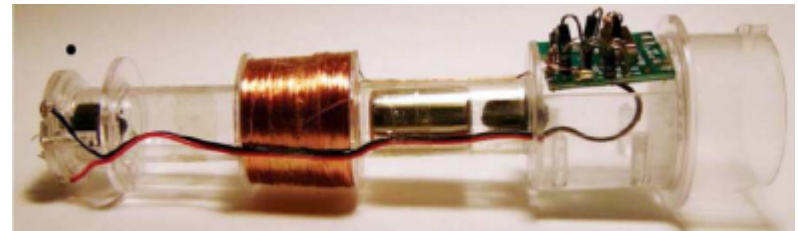
Simple principle:

- Shake the box
- Mass moves relative to the frame
- Transducer damps the motion and outputs electrical power
- Aim is to maximise power dissipation (conversion) in the damper (again, force distance integral)
- Damper can be electromagnetic, electrostatic or piezoelectric
- Can't just make damping force arbitrarily large as this limits mass travel



# Inertial Generator - Large Examples

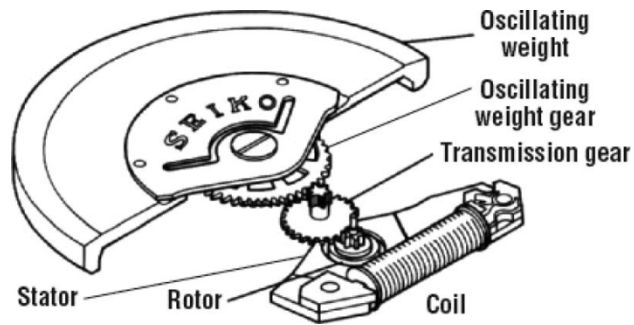
Larry Rome, Penn State



Harvesting torch

## Inertial Devices – smaller examples

- Capture energy from the environment and convert to an electrical form



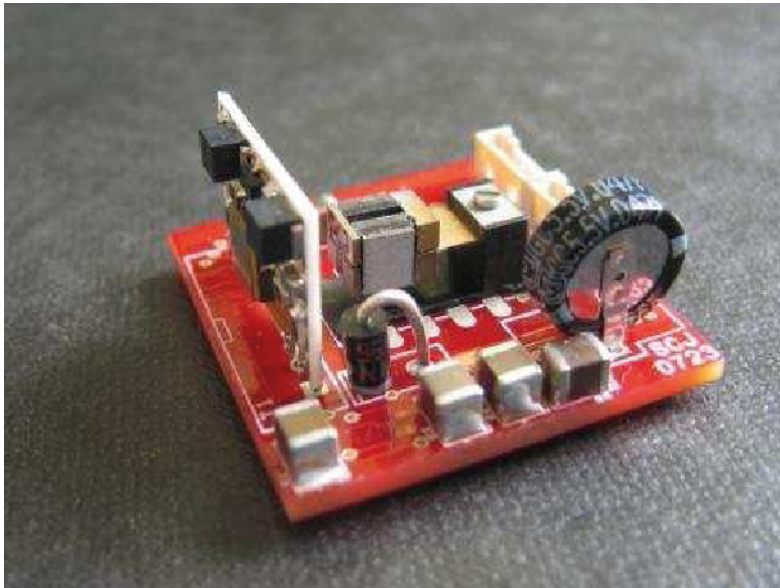
*Seiko kinetic watch generator*



- PMG17 from Perpetuum Ltd
- Resonant generator tuned to 100 or 120 Hz
- 55 mm diameter x 55 mm length
- 4.5 mW output power (rectified DC) at 0.1g acceleration

## Electromagnetic Transducers - example

- Vibes Generator (Steve Beeby and others)

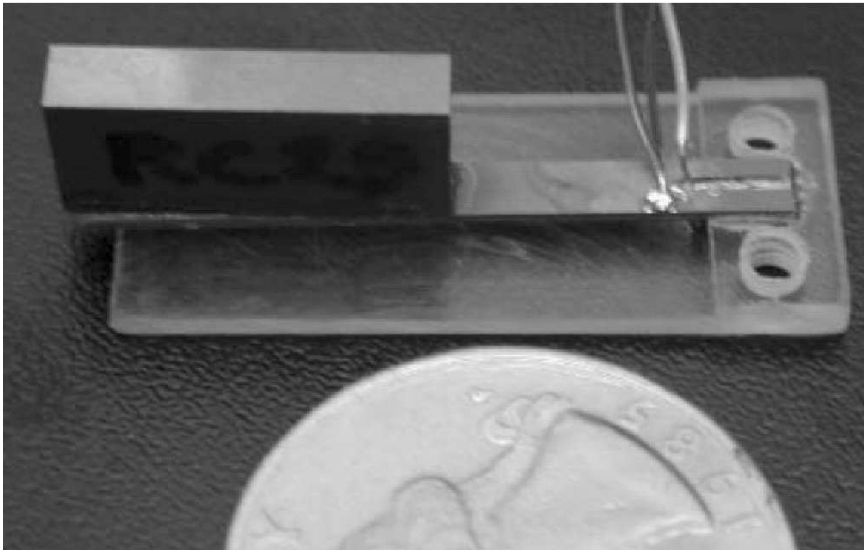


- 2800 turns on coil
- 50 Hz and 60 mg operation
- Output voltage around 700 mV RMS
- Output power 55  $\mu$ W

Self-powered sensor and transmitter demonstrated

## Piezoelectric Transducers - example

- UC Berkeley Generator (Shad Roundy and others)



- 120 Hz and 60 mg operation
- Input amplitude  $4.4 \mu\text{m}$
- Output power  $116 \mu\text{W}$

Self-powered sensor and transmitter  
demonstrated from similar device

## Electrostatic Transducers - example

- MIT Generator (Mur Miranda and others)



- Referred to as comb drive in MEMS community
- Constant  $V$  sliding approach
- Simulations show an expected  $8 \mu\text{W}$  from a  $2.5 \text{ kHz}$  input

## Summary of Overview

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- Large scale power generation uses direct-force type generation
- This is not suitable for harvesters because applications usually limit the generator to one attachment point
- This (as well as practical constraints) places fundamental limits on the power density of these systems
- Kinetic harvesters have been seen in some practical scenarios and are a still growing research topic

# Transducers Overview

*Performance limits of the three MEMS inertial energy generator transduction types*

*PD Mitcheson, EK Reilly, T Toh, PK Wright, EM Yeatman*

*Journal of Micromechanics and Microengineering 17 (9), S211*

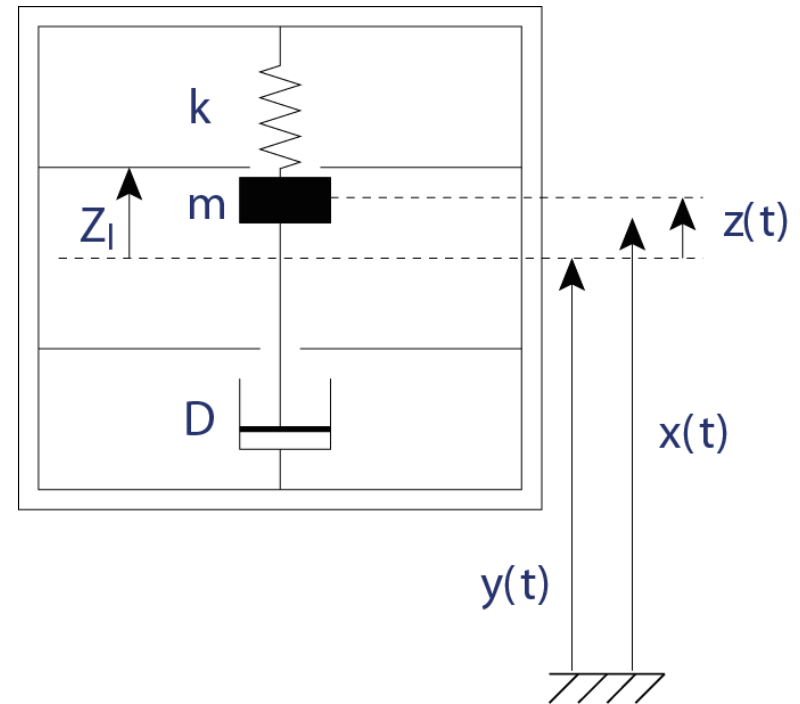
# Transducers

- There are essentially two phenomena that can be used to convert (relative) motion into electrical energy (or vice versa)
- Electromagnetic/electrodynamic force
- Normally used in macro scale devices (eg motors, power stations...)
- Electrostatic force
- Often found in MEMS
  
- The electromagnetic force can be implemented
  - with a coil and magnet
- The electrostatic force can be implemented in several ways
  - Moving plate capacitor
    - (dis)charged by an external circuit
    - Primed using an electret
  - A piezoelectric material

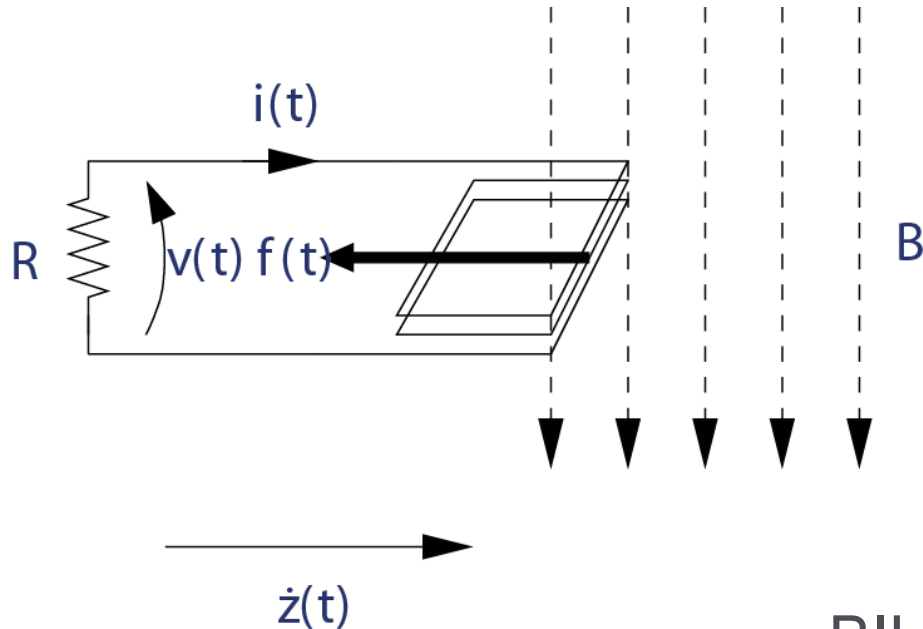


## Which is Best and Why?

- Surprisingly hard question to answer!
- Back to the mass in a box
- What value of damper should we choose to maximise power generation?
- What does this depend on?
- What should the damper characteristic be? Linear, non-linear?
- What are the practical limits?
- How difficult/efficient is each damper when connected to a circuit?



## Electromagnetic Transducers – basic theory



- Change of flux induces a voltage on the coil
- Current flows
- Lorentz force acts as *linear* damper

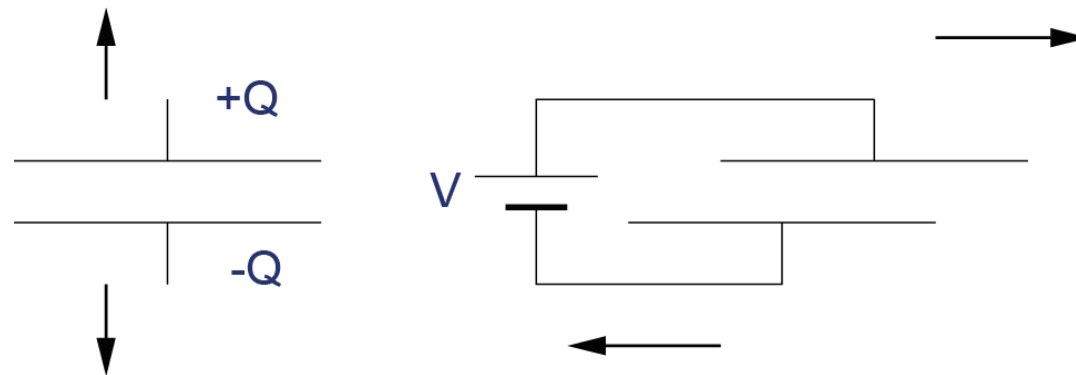
$$F(t) = \frac{(NBl)^2 \dot{z}(t)}{R}$$

- $BIL$  product from generated emf
- $N$  is number of coil turns and  $l$  is active coil length

*If we ignore coil inductance force is proportional to velocity*

## Electrostatic Transducers - options

- Easy to achieve a Coulomb force, and two methods available:
  - Changing separation of plates in constant charge and constant overlap
  - Changing overlap of plates at constant voltage and constant separation



Energy generated is Force  $\times$  Distance

## Electrostatic Transducers - equations

- Changing separation at constant Q:

$$F = \frac{1}{2} \frac{Q^2}{\epsilon A}$$

$A$  is plate area

- Changing overlap at constant V:

$$F = \frac{1}{2} \frac{V^2 \epsilon w}{d}$$

$w$  is plate width and  $d$  the separation

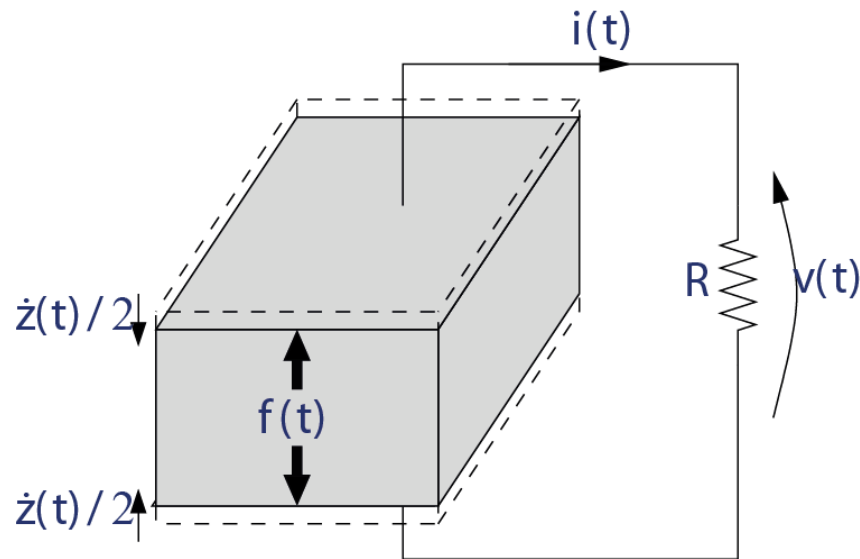
- In both cases  $F$  is constant (Coulomb)

## Constant Q Operation

- Capacitor must be precharged at  $C_{\max}$  to optimal voltage (to give optimal damping)
- Plates separate to give  $C_{\min}$  under constant Q
- Voltage on plates rises
- Energy generated is:

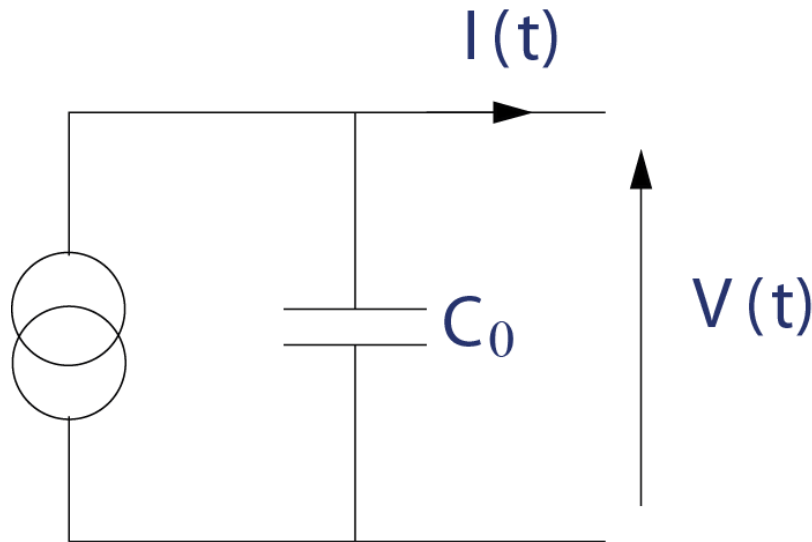
$$E = \frac{1}{2} Q^2 \left( \frac{1}{C_{\min}} - \frac{1}{C_{\max}} \right)$$

## Piezoelectric Transducers - Principle



- When the piezoelectric is strained, a current is generated
- Some of the charge ends up on the internal capacitance
- Some can flow in an external load

## Piezoelectric Transducers - equations



$$F(t) = K_{PE}z(t) + \alpha V(t)$$

$$I(t) = \alpha \dot{z}(t) - C_0 \dot{V}(t)$$

$F(t)$  is the force on the transducer,  $C_0$  the internal piezo capacitance,  $V(t)$  the terminal voltage,  $z(t)$  the displacement and  $\alpha$  and  $K_{PE}$  are material constants

## Piezoelectric Transducers - behaviour

- For a resistive load the transducer force is:

$$F(s) = Z(s) \left( K_{PE} + \frac{s\alpha^2 R}{1 + sC_0 R} \right)$$

- This can be written as a constant valued spring constant ( $K_{PE}$ ) plus a frequency dependent high pass term:

$$F_{HP}(j\omega) = Z(j\omega) \left( \underbrace{\frac{j\omega\alpha^2 R}{1 + (\omega C_0 R)^2}}_{\text{damper}} + \underbrace{\frac{\omega^2\alpha^2 C_0 R^2}{1 + (\omega C_0 R)^2}}_{\text{spring}} \right)$$



## Piezoelectric Transducer - explained

- We can therefore think of the piezoelectric transducer with a resistive load as presenting a spring constant and a damping term
- Both of these are frequency sensitive
- Nonetheless, at a given frequency, the damping can be calculated and the system behaves linearly (i.e. generates no harmonics)
- Damping is maximised ( $dF/dR=0$ ) when:

$$R = \frac{1}{\omega C_0}$$

## Practical Aspects

- There are limits on the maximum forces that can be developed for these transducers
  - There are maximum voltage limits on the piezoelectric and electrostatic devices due to either breakdown of the dielectric medium, or the attached circuit components
  - There are limits on the current that can flow in the electromagnetic transducer due to coil resistance
  - There are limits on damping in the piezoelectric device due to the output capacitance (i.e. not all the displaced charge can be forced through a chosen load)
- The circuit requirements have a significant effect on system performance

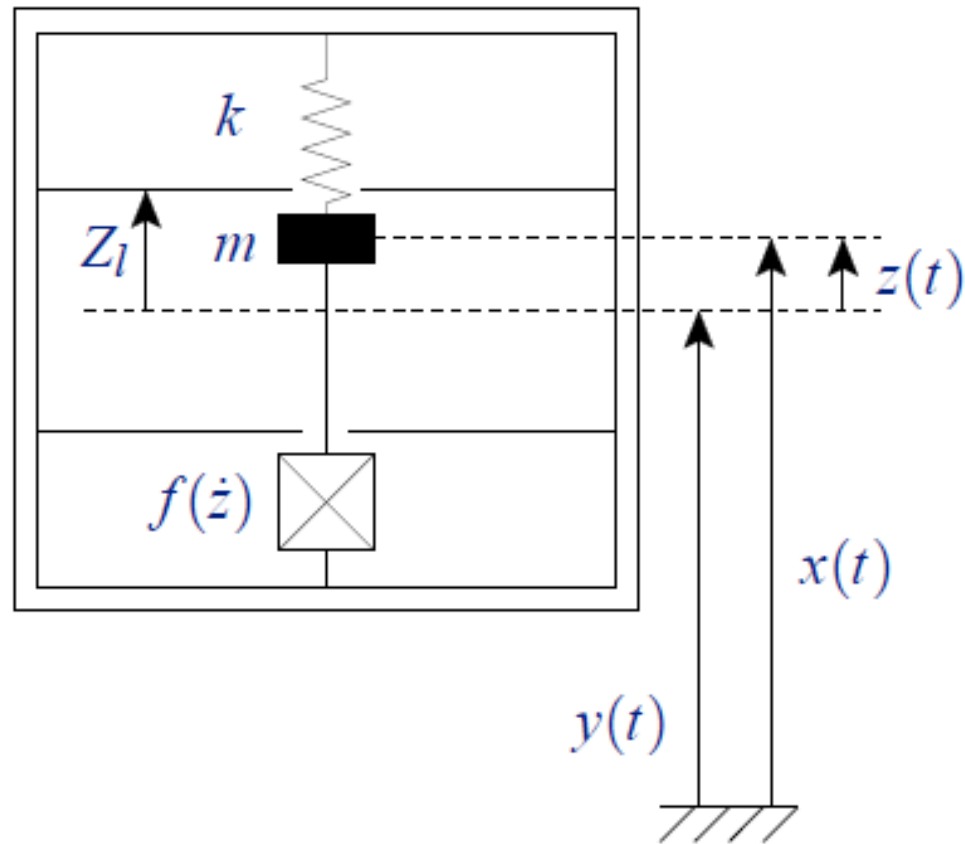
## Summary of Transducers

- 2 phenomena
  - Electromagnetic or electrodynamic
  - Electrostatic
- 3 common implementations
  - Magnetic and coil
  - Moving plate capacitor
  - Piezoelectric
- Each has a different velocity-force characteristic (which also depends on the connected circuit element)
- Practical constraints on each type limit the magnitude of the damping

# Fundamental Analysis and Optimisation

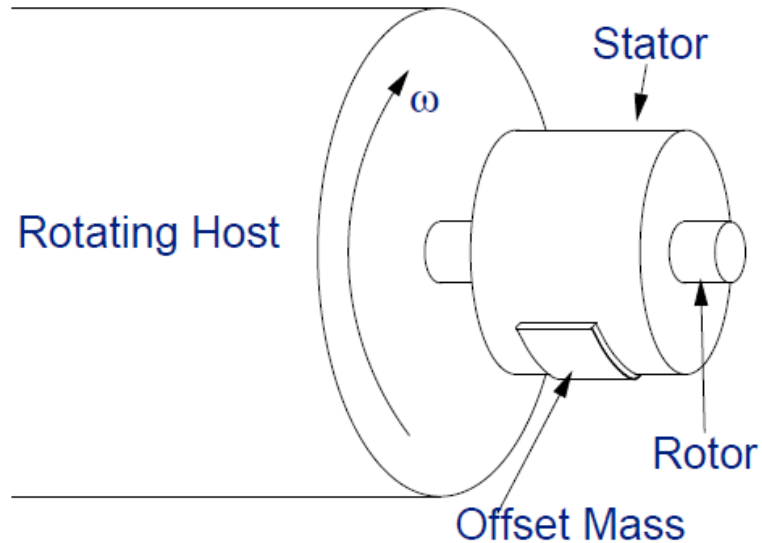
*Architectures for vibration-driven micropower generators*  
*PD Mitcheson, TC Green, EM Yeatman, AS Holmes*  
*Microelectromechanical Systems, Journal of 13 (3), 429-440*

## Optimise the mass in a box model...



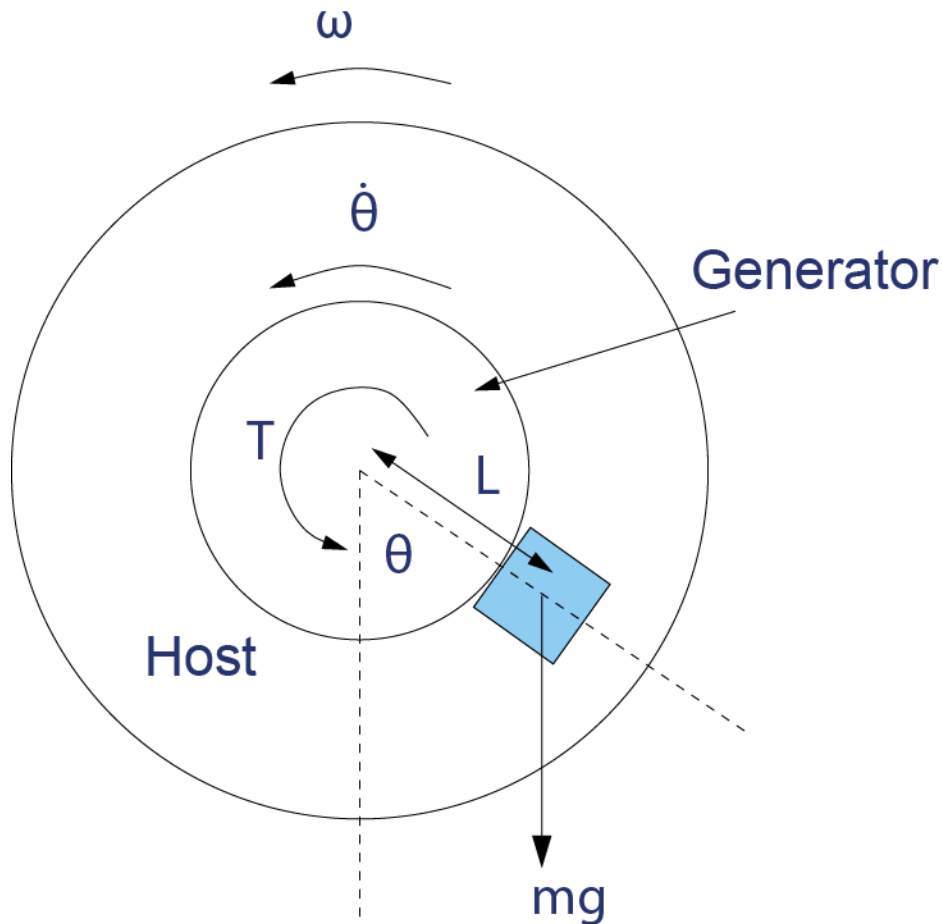
*How do we maximise the power dissipated in the damper?*

## Simple Example as a Warm-Up!



- Host structure rotates
- Gravitational torque on offset mass holds stator in place
- Power generated is produced of force and angular velocity

## Rotational Micro-Generators – Formula



- Power is simply Torque  $\times$  angular velocity

$$P = mgL \sin \theta$$

*And to optimise...*

## Rotational Generator Optimised Power

- So the limit on power generation in the rotational case is:

$$P_{\max} = \omega mgL$$

- Requires motor torque to be held at:

$$T_{\text{opt}} = mgL$$

- Overestimates power at low speeds with electromagnetic implementation
- This is because winding resistance ignored in basic model



## Design Choices

- We have seen there are 3 ways of implementing the damper. What are their characteristics?
  - Electromagnetic - the force will likely be proportional to velocity.
  - Piezoelectric - force could be proportional to velocity
  - Electrostatic - Coulomb force (like sliding friction)
- And we have other design choices to make for our mass in a box
  - Resonant or non-resonant?
  - Aspect ratio, materials, etc

## Inertial Generator Architectures

<i>Architecture</i>	<i>Damper</i>	<i>Spring</i>	<i>Damping Force</i>
Resonant	El. Mag/Piezo	$k = m\omega_n^2$	$f(t) = \dot{z} \times D$
Resonant	Electrostatic	$k = m\omega_n^2$	$f(t) = -\text{sgn}(\dot{z}) \times F$
Non-resonant	El. Mag/Piezo	$k = 0$	$f(t) = \dot{z} \times D$
Non-resonant	Electrostatic	$k = 0$	$f(t) = -\text{sgn}(\dot{z}) \times F$

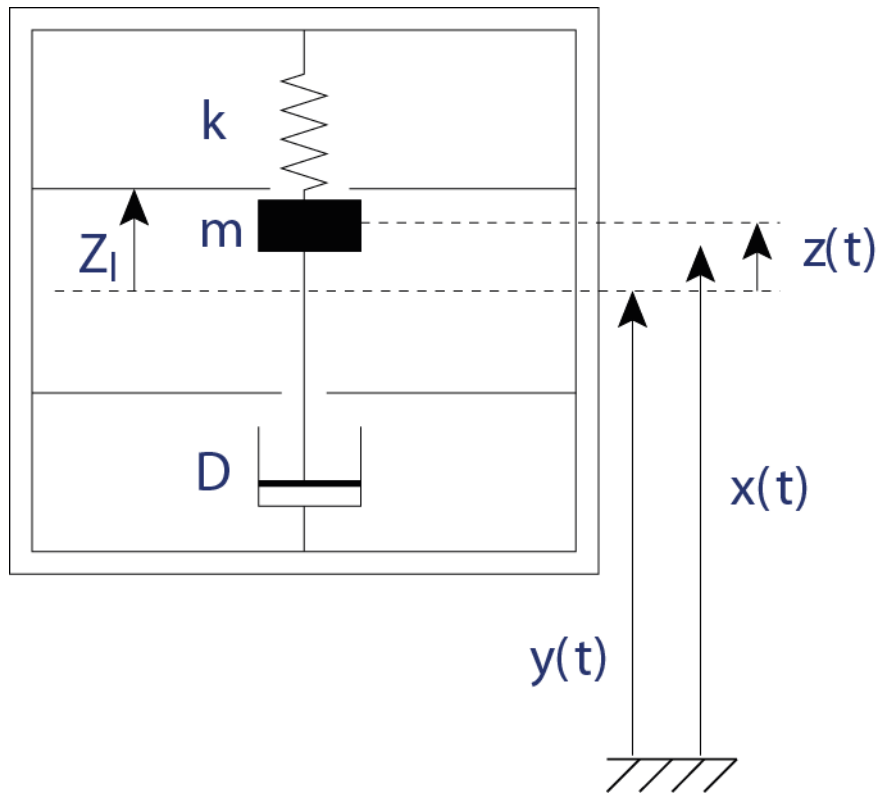
- Back of envelope calculation shows that the non-resonant velocity-damped generator is not practical on a small scale
- It would require more damping force than can be achieved with practical magnet/coil arrangements

## Inertial Generator Architectures

- Three practical architectures
  - Velocity-Damped Resonant Generator (VDRG):  
Generator with a tuned mass-spring system and a linear damper
  - Coulomb-Damped Resonant Generator (CDRG):  
Generator with a tuned system and a non-linear damper
  - Coulomb-Force Parametric Generator (CFPG):  
System without a spring and with a non-linear damper

*Which is the best? When? Let's analyse them...*

## Velocity-Damped Resonant Generator



- Newton's 2nd Law:

$$m\ddot{z}(t) = -kz(t) - D\dot{z}(t) - m\ddot{y}(t)$$

- Transfer function:

$$\frac{Z(s)}{Y(s)} = \frac{-s^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

- Where:

$$\zeta = \frac{D}{2m\omega_n}, \omega_n = \sqrt{\frac{k}{m}}$$

## Velocity-Damped Resonant Generator

- The frequency response:

$$\frac{Z_0}{Y_0} = \frac{\omega_c^2}{\sqrt{(1 - \omega_c^2)^2 + (2\xi\omega_c)^2}}$$

- Energy per cycle:

$$E = 2D \int_{-Z_0}^{Z_0} \dot{z} dz$$

- And thus:

$$P = \frac{\xi\omega_c^3 Y_0^2 \omega^3 m}{[1 - \omega_c^2]^2 + [2\xi\omega_c]^2}$$

## VDRG – Maximising Power Density

- Find the optimal damping factor:

$$\frac{dP}{d\xi} = 0$$

- This gives:

$$\xi_{opt} = \frac{1}{2\omega_c} \sqrt{\omega_c^4 - 2\omega_c^2 + 1}$$

- But need to consider the constraints:
  - The limitation on  $Z_l$
  - The maximum value of physical constants

*We want to find the best value of  $\zeta$  under all conditions*

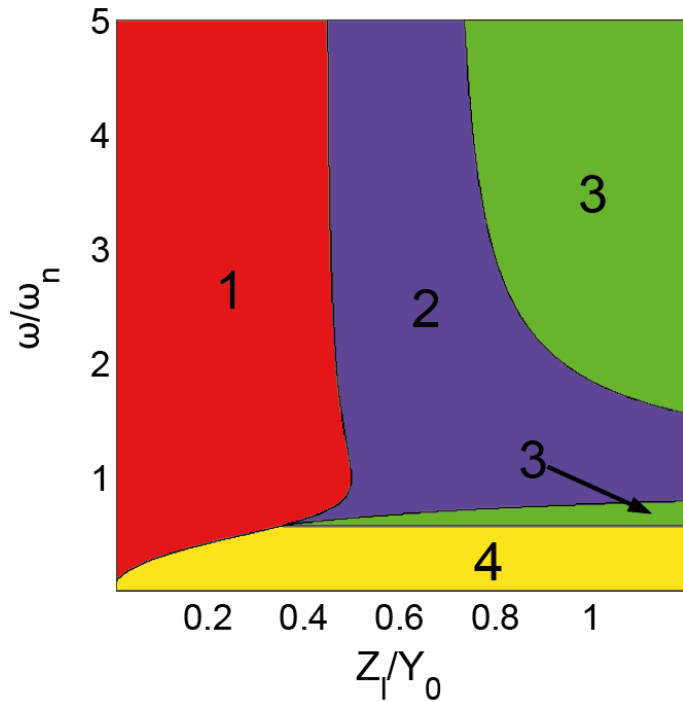
## VDRG – Maximising Power Density

- The expression for maximum power density (under all conditions) can always be written as a function of:
  - The ratio of  $\omega/\omega_n$
  - The ratio of  $Z_l/Y_0$
- And can be normalised to:
  - The value of the proof mass
  - The cube of the input motion frequency
  - The square of the input motion amplitude

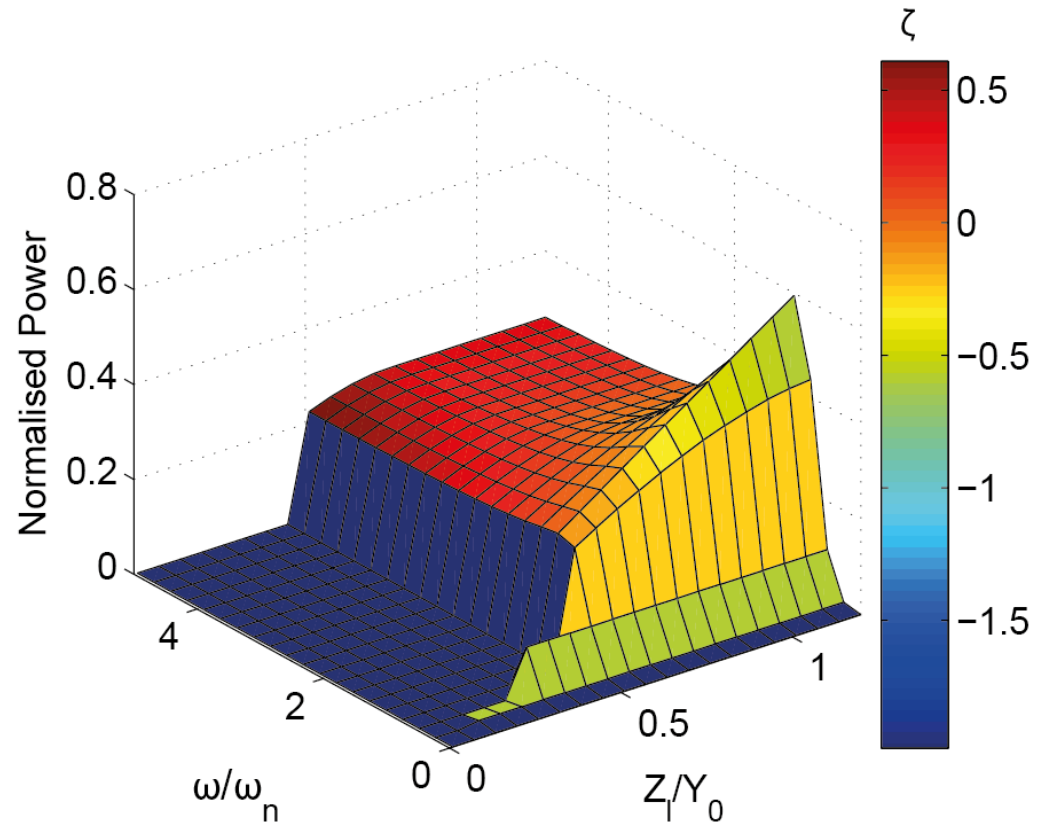
*We can now plot an optimal performance surface...*

## VDRG Optimal Performance

$m=0.5$  g,  $B=1$  T,  $N=10$ ,  $R=1$   $\Omega$ ,  $\omega=10$  rads/s,  $A=1$  cm<sup>2</sup>



Operating chart



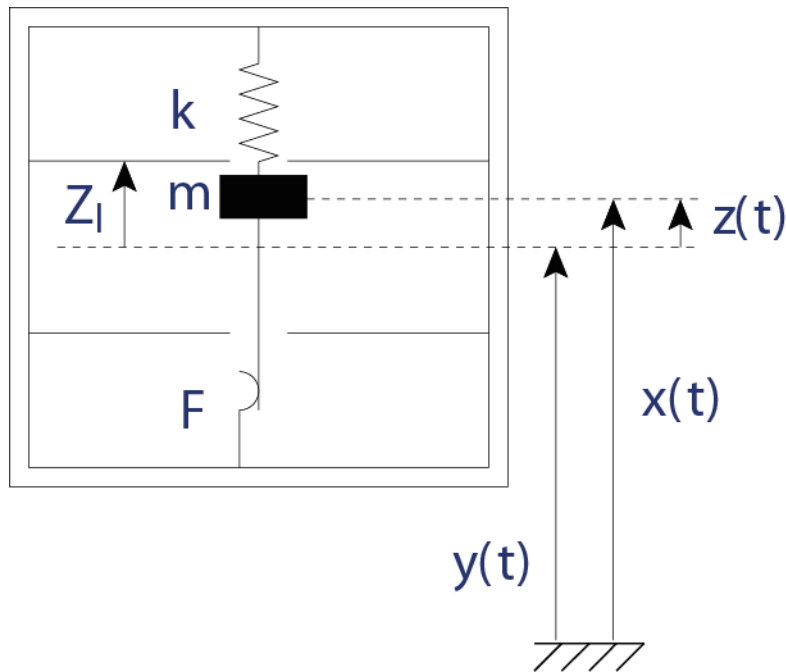
Maximum power



## VDRG Operating Regions

- With reference to the operating chart on the previous page
  1. Device unable to operate, the required  $\zeta$  to meet the displacement constraint being greater than the system can achieve.
  2. Power limited by  $Z_l$  - device operating at displacement limit
  3. Device operating optimally for the given value of  $\omega_c$ .
  4. More power could be generated if the damping factor could be increased above the value of  $\zeta_{\max}$ .

## Coulomb-Damped Resonant Generator



- Newton's 2nd Law:

$$m\ddot{z}(t) = -kz(t) - F \times \text{sgn}(\dot{z}(t)) - m\ddot{y}(t)$$

- Describing function:

$$\frac{Z_0}{Y_0} = \omega_c^2 \left[ \frac{1}{(1 - \omega_c^2)^2} - \left( \frac{F}{mY_0\omega^2\omega_c} U \right)^2 \right]^{\frac{1}{2}}$$

- Where:

$$U = \frac{\sin\left(\frac{\pi}{\omega_c}\right)}{\left[1 + \cos\left(\frac{\pi}{\omega_c}\right)\right]}$$

## Coulomb-Damped Resonant Generator

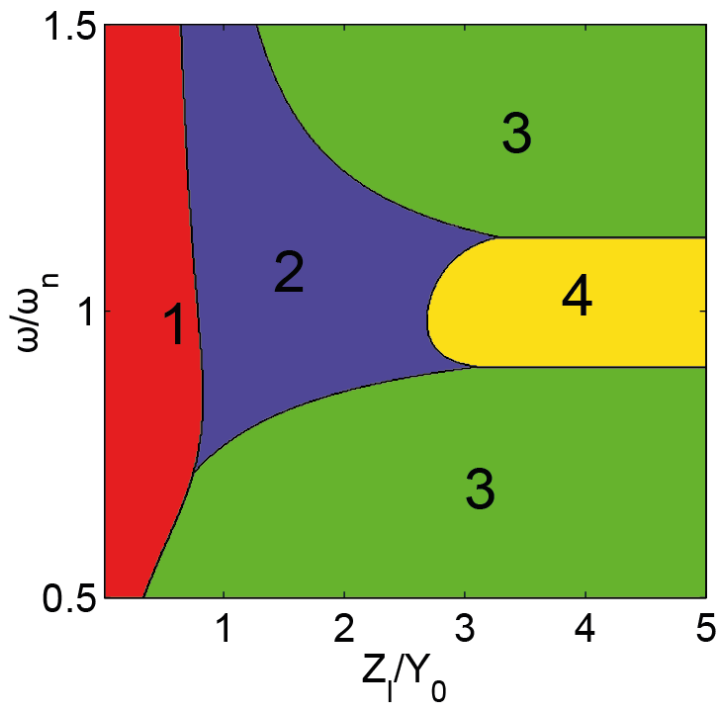
- The energy dissipated is given by the force-distance product, and thus the power is:

$$P = \frac{4Y_0 F \omega \omega_c^2}{2\pi} \left[ \frac{1}{(1 - \omega_c^2)^2} - \left( \frac{F}{mY_0 \omega^2 \omega_c} U \right)^2 \right]^{\frac{1}{2}}$$

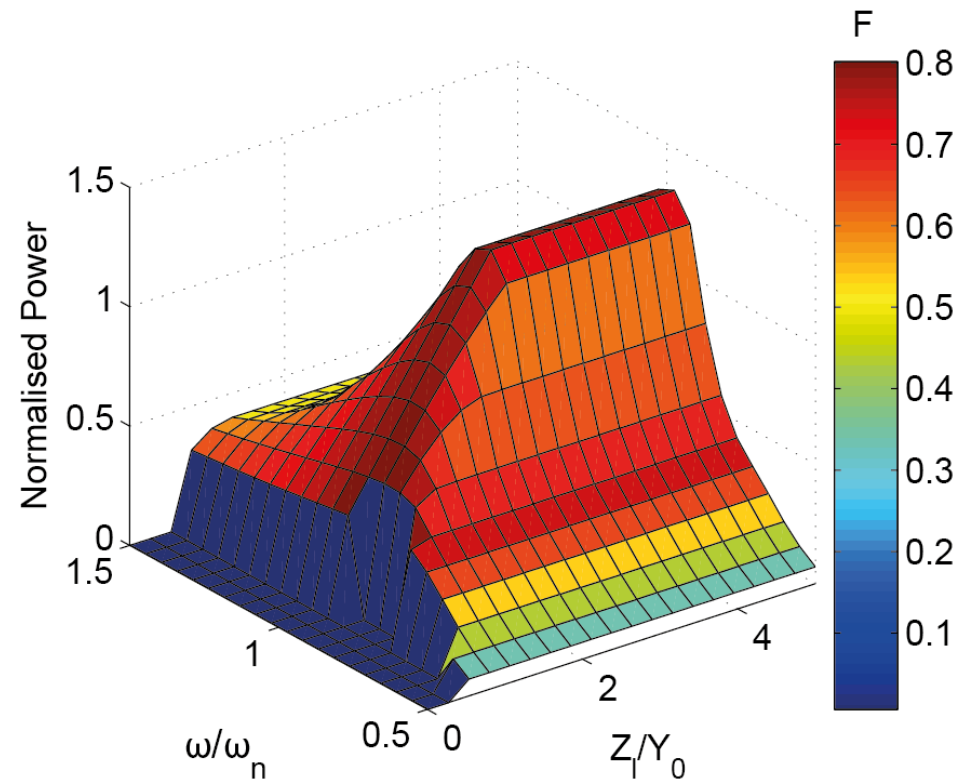
- We may now find the optimal damping force, and the operating regions as we did for the VDRG...

## CDRG Optimal Performance

$m=1$  g,  $Y_0=1$  mm,  $\omega=20\pi$ ,  $A=1$  cm<sup>2</sup>,  $V_{\max}=450$  V



Operating chart

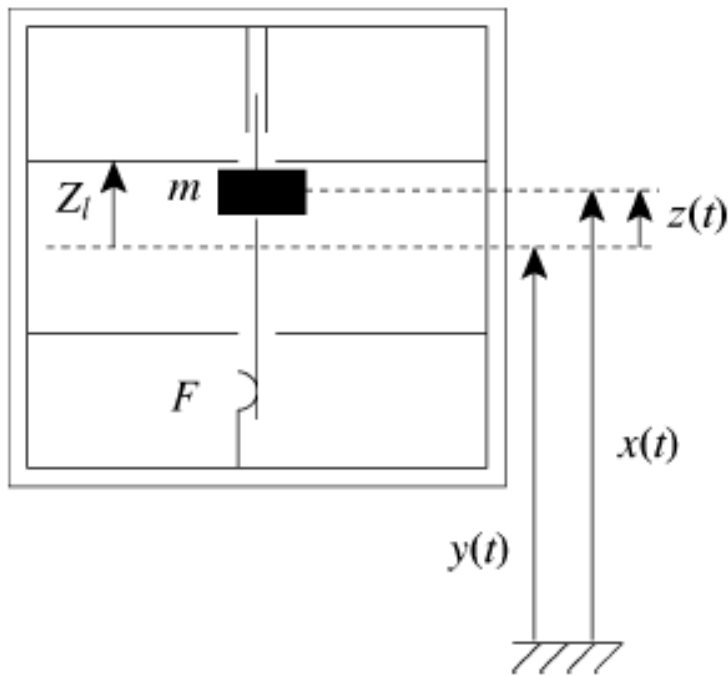


Maximum power

## CDRG Operating Regions

- With reference to the operating chart on the previous page
  1. Device can't operate without stops in motion (non-linear damper can cause stop-start motion)
  2. Power limited by  $Z_l$  - device operating at displacement limit
  3. Device operating optimally for the value of  $\omega/\omega_n$
  4. Device operation limited by maximum voltage

## Coulomb-Force Parametric Generator



- Maximum inertial force on mass:

$$mY_0\omega^2$$

- Thus energy per stroke is the force distance product:

$$E = \beta m Y_0 \omega^2 Z_0$$

- $\beta$  is the break-away factor and is less than 1 (in order for the mass to move some distance the force must be a fraction of the maximum inertial force)

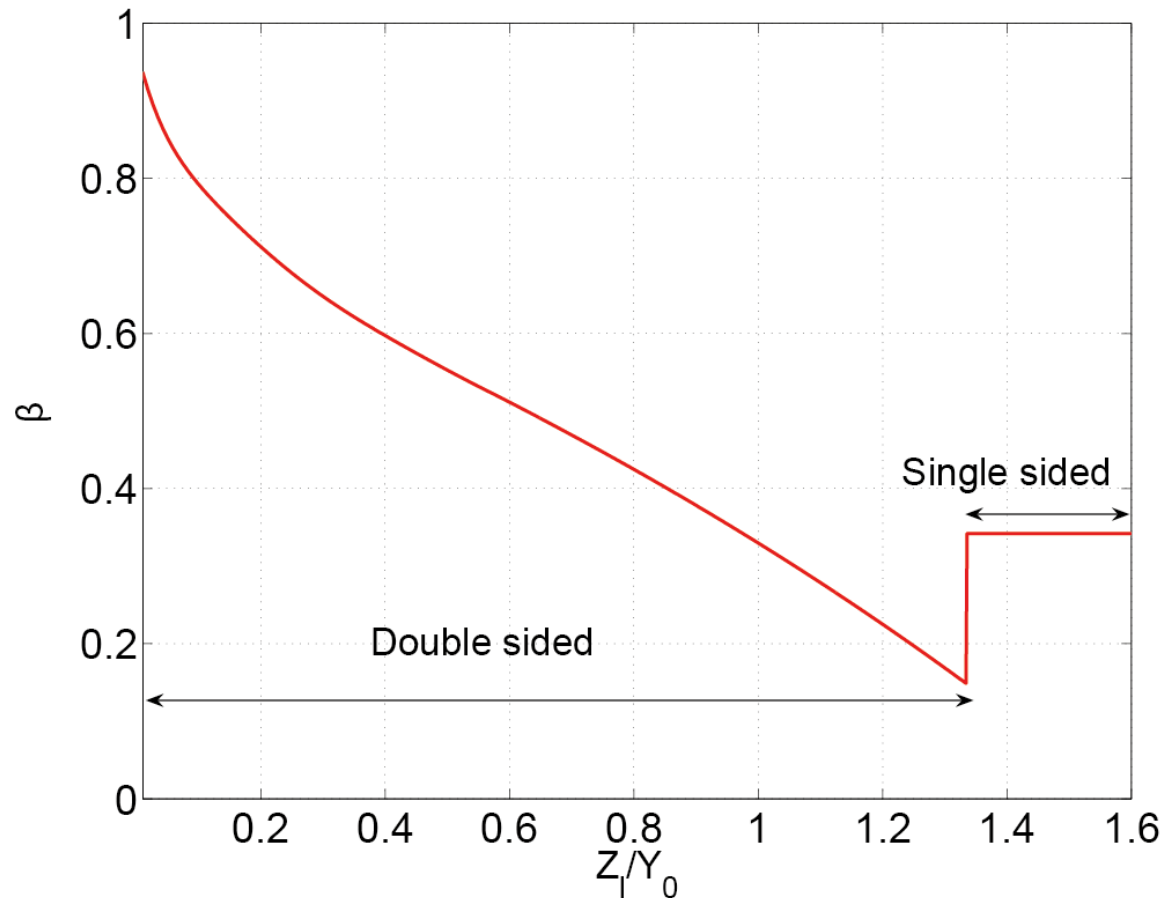
## Coulomb-Force Parametric Generator

- Thus, the power is given by:

$$P = 2\beta \left( \frac{Z_0}{Y_0} \right) \frac{Y_0^2 \omega^3 m}{\pi}$$

- There is an optimal  $\beta$
- Must maximise  $\beta Z_0$  product
- Maximum value of  $Z_0$  is  $Z_l$
- Can't solve for  $\beta_{opt}$  analytically...

# Coulomb-Force Parametric Generator



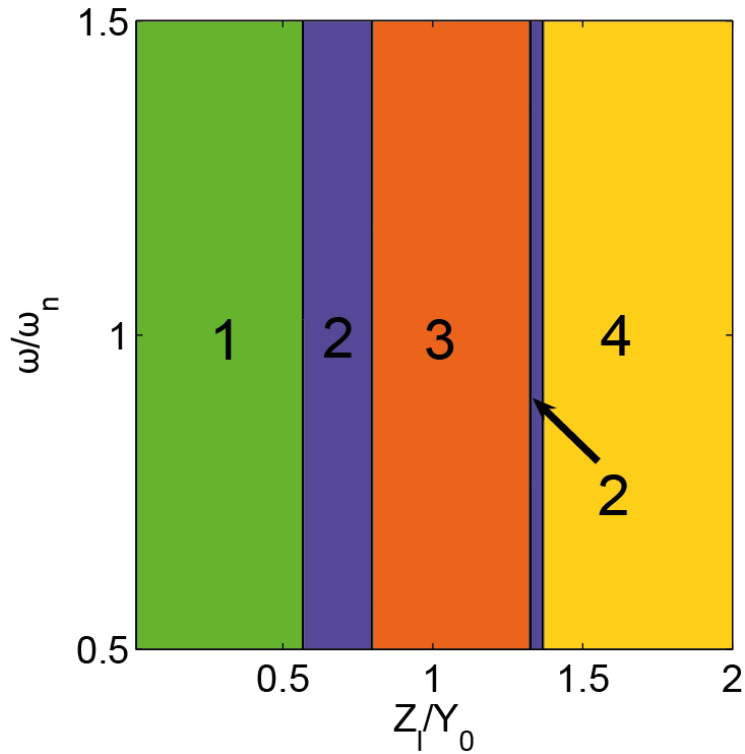
- $\beta_{\text{opt}}$  calculated numerically from simulation
- Polynomial fitted to the results (shown)
- Change from double to single sided operation

- Now we know  $\beta$  we can plot the optimal performance...

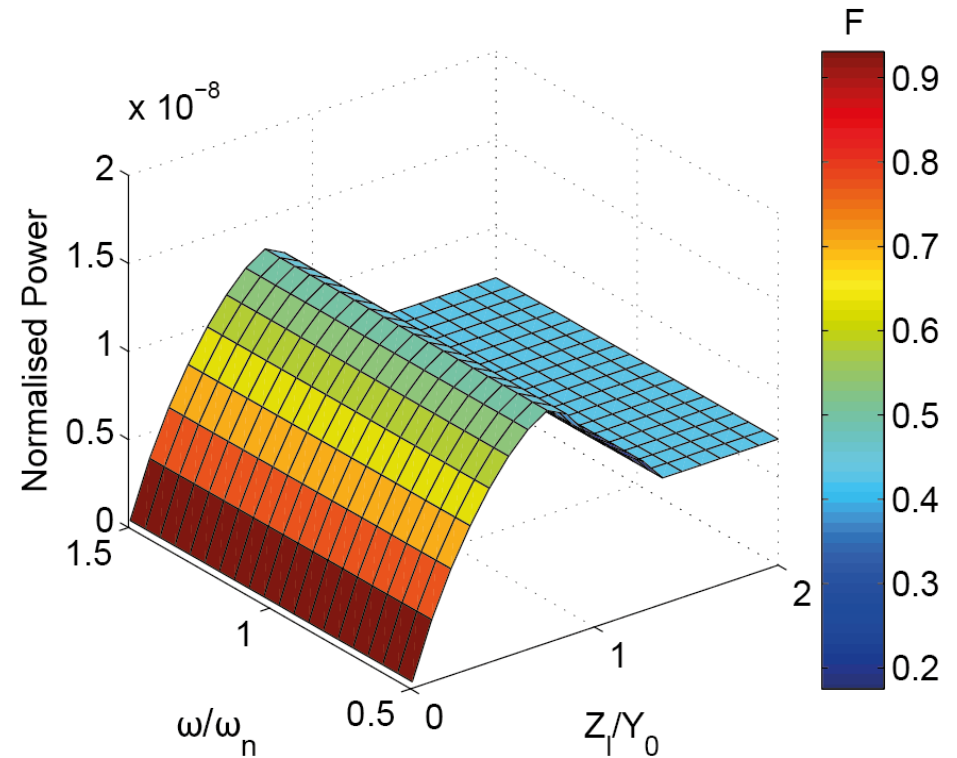


## CFPG Optimal Performance

$m=1$  g,  $Y_0=0,5$  mm,  $\omega=2\pi$ ,  $A=1$  cm<sup>2</sup>,  $V_{\max}=110$  V



Operating chart



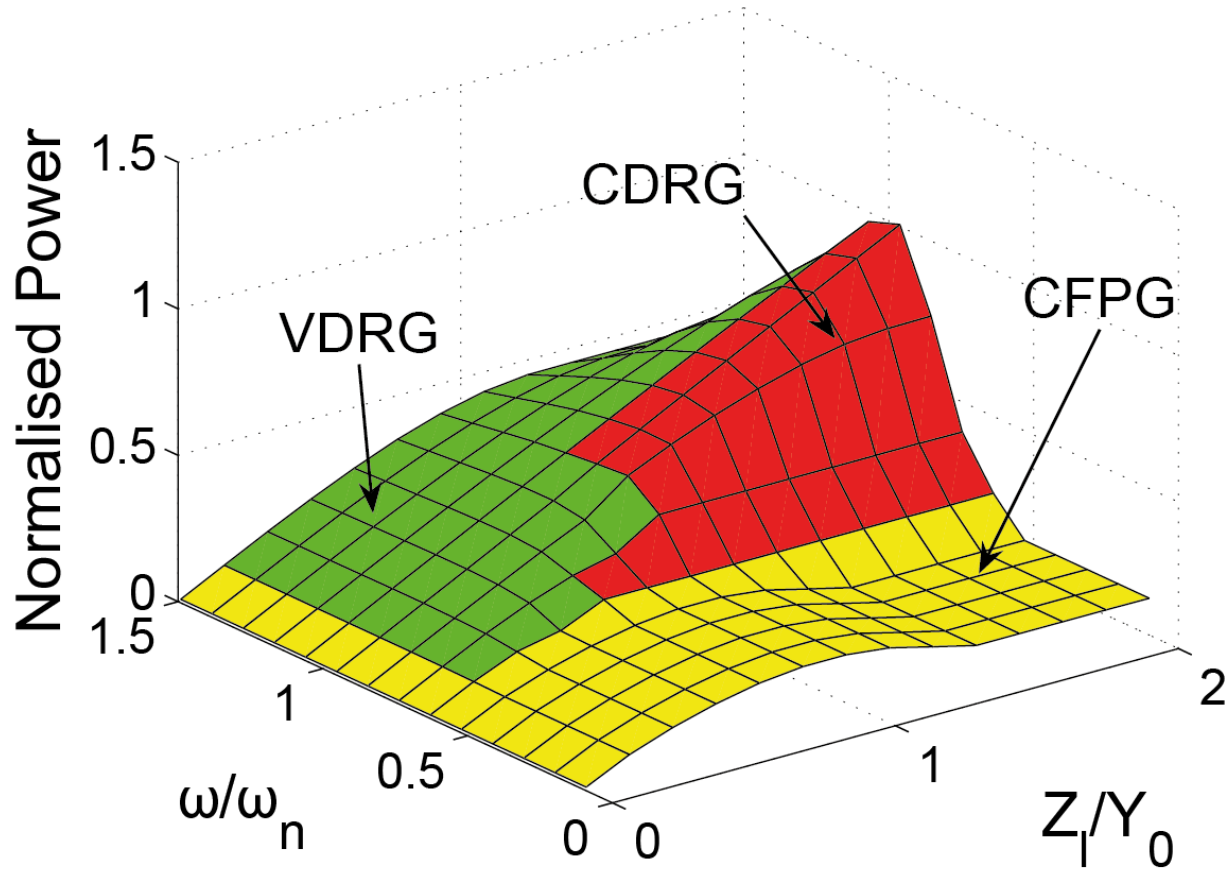
Maximum power

## CFPG Operating Regions

- With reference to the operating chart on the previous page
  1. Optimal double-sided operation
  2.  $\beta$  reduced to allow double sided operation
  3. Device in voltage limit – more power could be generated if the device was allowed to operate at over 110 V
  4. Single sided mode and device still in voltage limit

*We can now compare the three architectures...*

## Architecture Comparison



Maximum performance under varying operating conditions

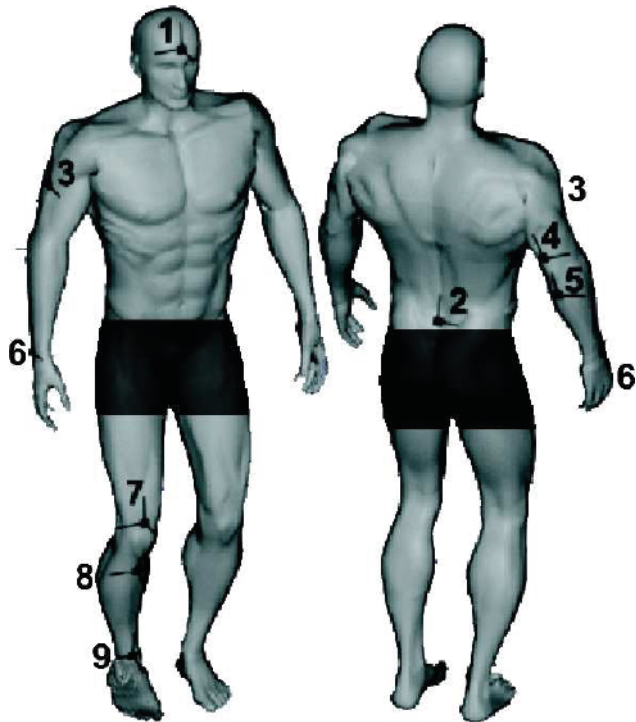
## Architecture Comparison

- Resonance only useful when  $Z_1/Y_0$  is large
- Very little difference between the VDRG and CDRG – choice mainly down to implementation and scaling
- CFPG better when  $Z_1/Y_0$  small – implanted devices powered from human body motion
- The CFPG doesn't have a resonant system to be tuned

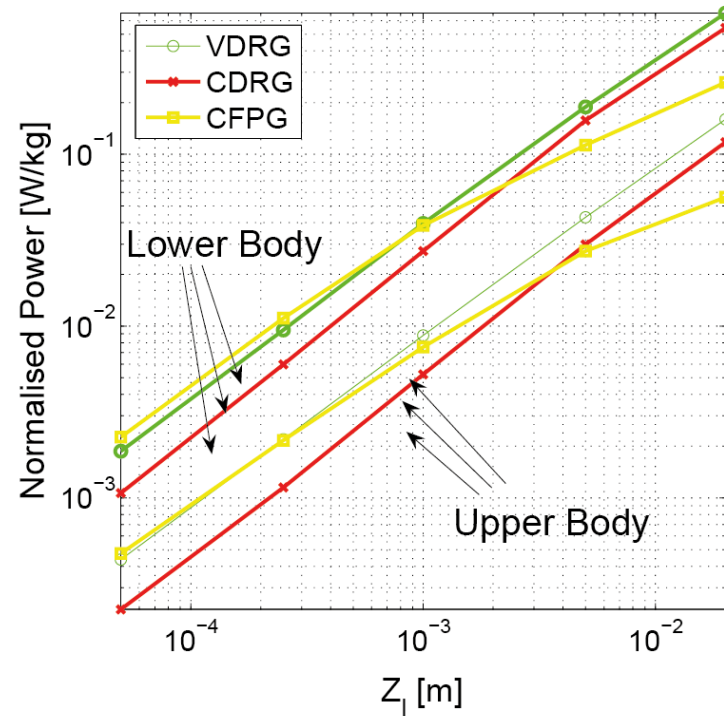
*But how do the generators perform on signals more complex than single sinusoids?*

# Human Powered Generators

Power from non-sinusoidal motion [8]



Sensor positions



Performance comparison

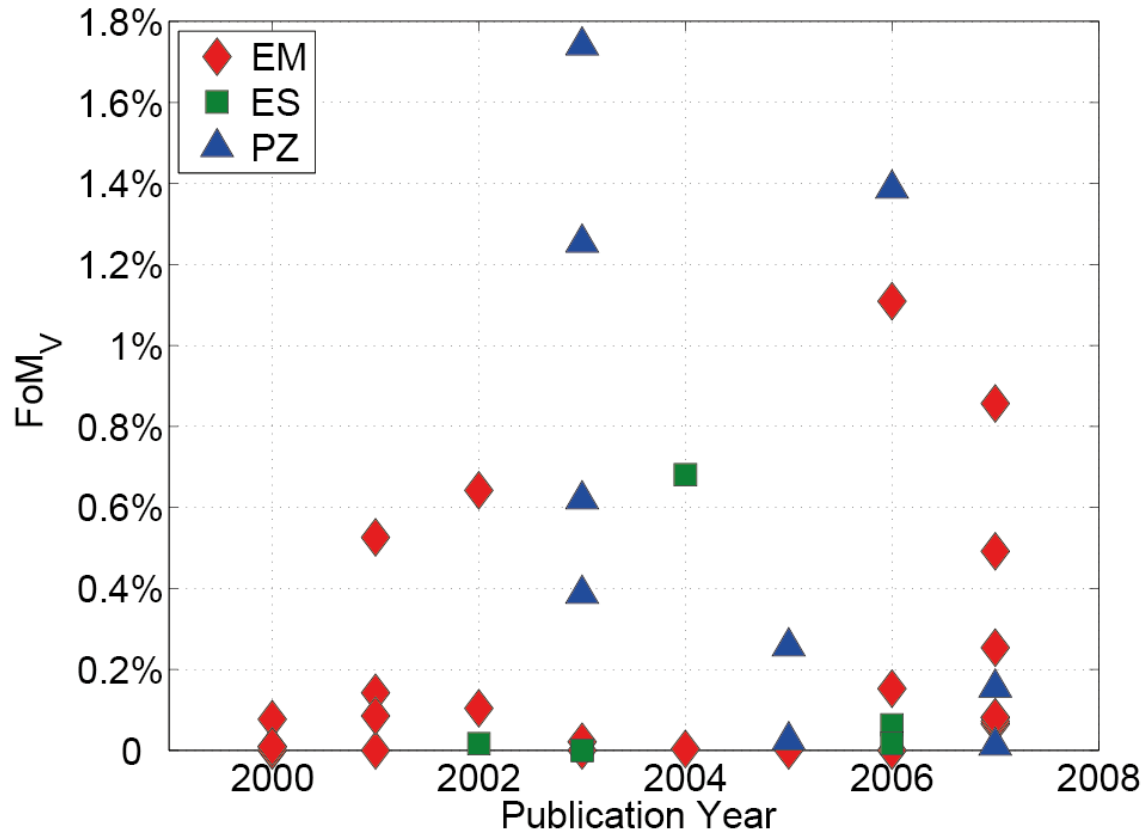
*This work was in collaboration with T. von Büren at ETH Zurich*

## Effectiveness of Microgenerators

- The efficiency is actually an effectiveness
- Many people have suggested different formulae
- Fights can break out whilst discussing this!
- I would argue there is only sensible metric...
- A figure of 100% on this scale is the absolute best you can do
- We call this metric Volume Figure of Merit

# Effectiveness of Inertial Microgenerators

$$FoM_V = \frac{\text{Useful Power Output}}{\frac{1}{16} Y_0 \rho_{Au} \text{Vol}^{\frac{4}{3}} \omega^3}$$



- Volume Figure of Merit
- Lots of room to improve!
- Not enough data yet on rotational case

## Summary of Harvester Fundamental Analysis

- We analysed the limits of power conversion using an inertial generator mass in a box model
- Power is maximised at resonance
- Power is proportional to the proof mass
- Power is proportional to the internal displacement
- Different damper types perform slightly differently but when optimised the limits are the same
- Resonance is useful when the device is larger than around 10 times the driving displacement amplitude
- For these systems power can always be normalised to  $Y_0^2 \omega^3 m$



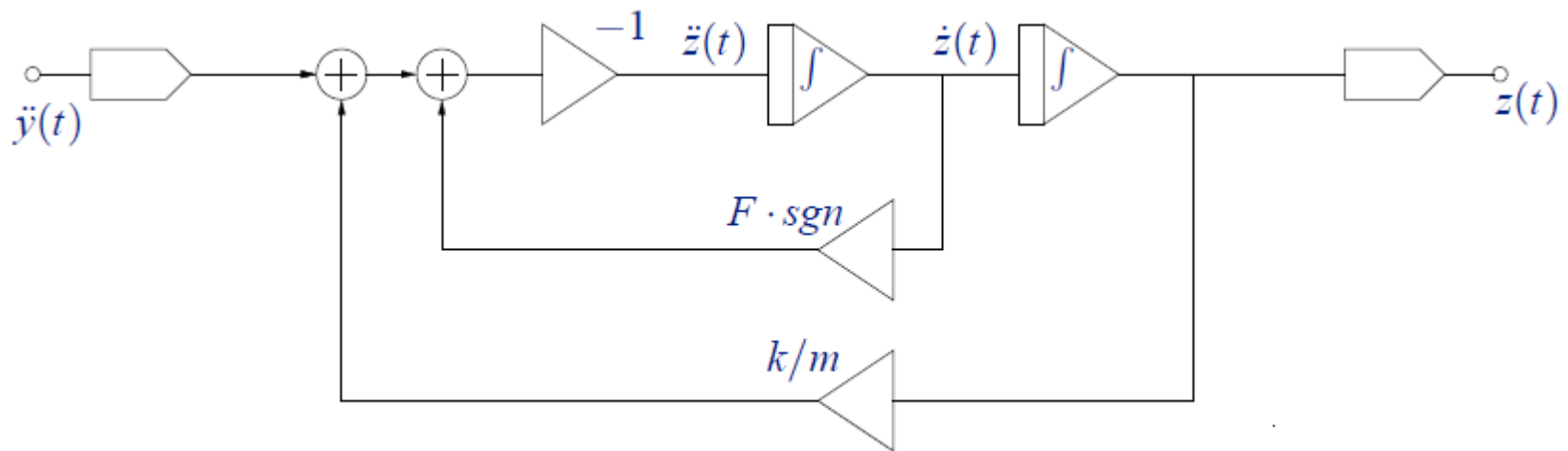
# **System Modelling and Simulation Approaches**

## Modelling Overview

- All our analysis so far has involved closed form algebra
- The damper characteristic is critical to the analysis
- Takes into account the coupling between mechanical and electrical
- “Simple” to analyse when the load is linear or switched on an electrostatic (because the damping characteristic is known)
- What type of model could we use for a complete electromechanical system analysis and simulation?
- The right model can give very good insights...

## One Potential Modelling Strategy...

- Normal type of simulation diagram used in Simulink

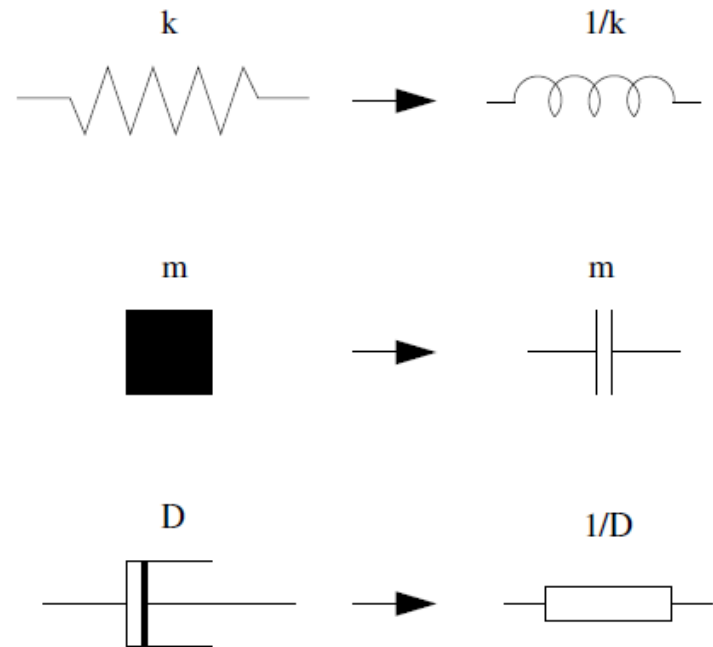


- This is a mixed electromechanical system model of a CDRG
- Probably also should include fluids effects, careful fields analysis too
- Allows time domain simulation... but does it give any real insight?

## Electrical/Mechanical Analogy

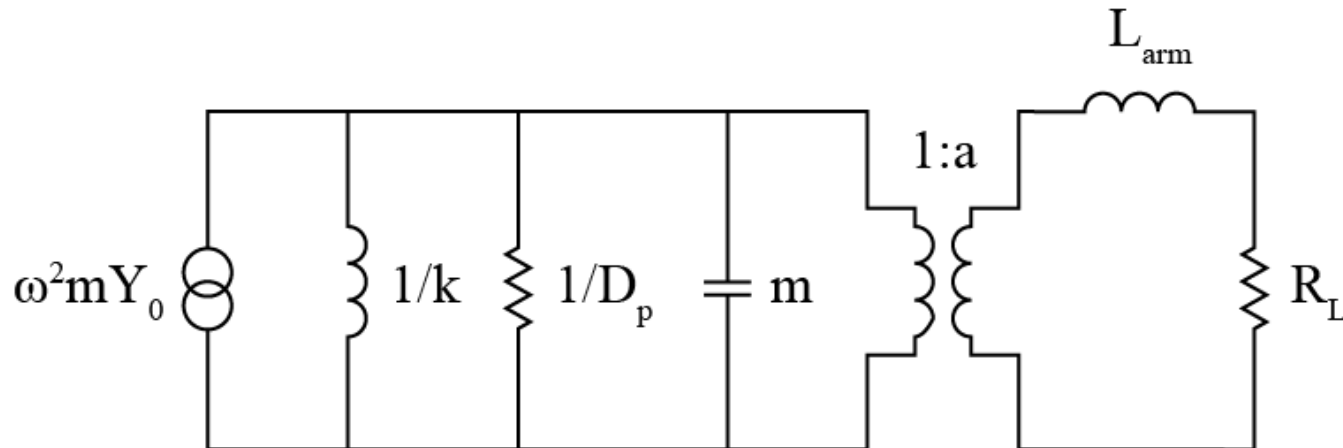
- Rather useful is the analogy between Newton's second law and Kirchoff's voltage law.
- The equations which describe a mass-spring-damper system are identical in form to those describing an RLC oscillator
- The mapping can be done using one of two conventions,
  - $f \rightarrow V$ , (flow, or velocity, corresponds to voltage)
  - $e \rightarrow V$  (effort, or force, corresponds to voltage)

*f*->*V* mappings



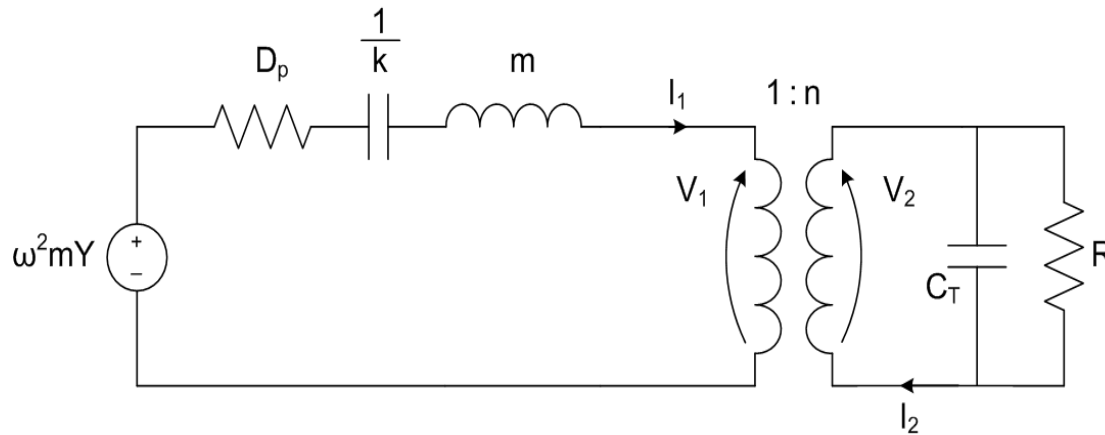
## Electromagnetic Harvester Model (f->V)

- Immediately you can see what happens when you operate at, or away, from resonance.
- You can also see what you need to do to maximise power with parasitic damping present



## Piezoelectric Harvester Model (e->V)

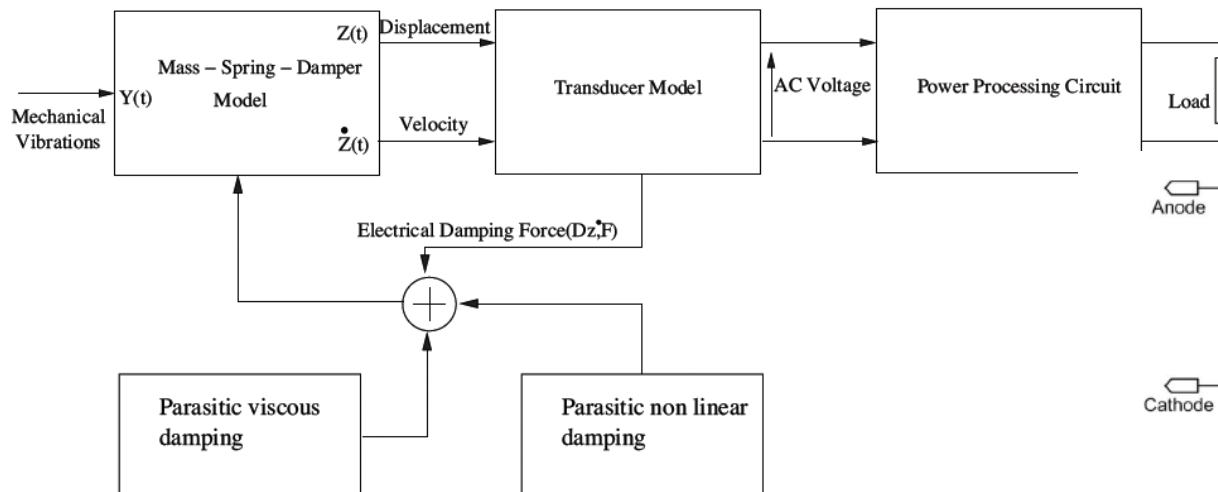
- More sophisticated to model the interaction between the load and the generator



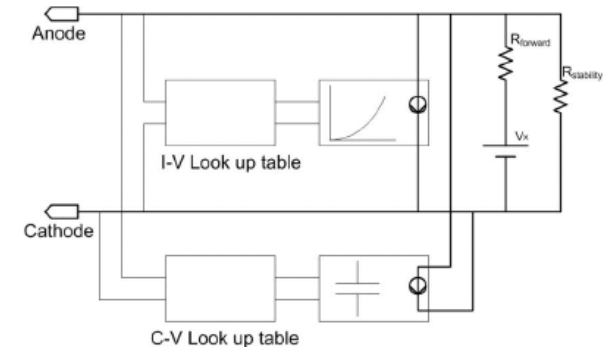
- This can be modelled in SPICE and can allow the load circuit to be modelled with good device models

## More Sophisticated Models

- Include the non-linear mechanical components (mass limited travel, spring hardening)
- Include custom semiconductor device models
- All done in SPICE – Imperial College Energy Harvesting Toolkit (ICES)

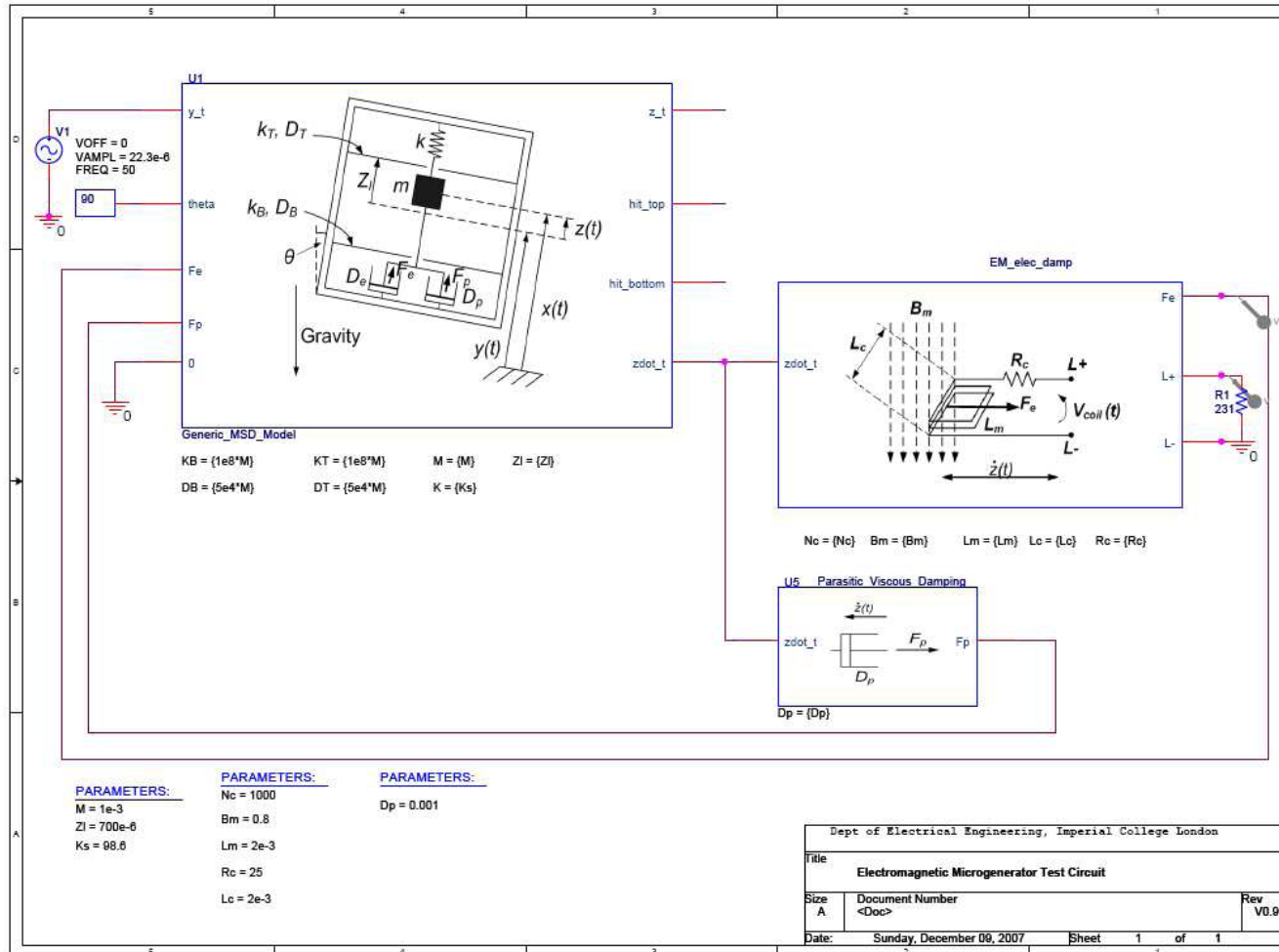


*Detailed  
Mechanics*



*Custom device  
model*

# ICES Toolkit





# **Transducers and Power interface Requirements**

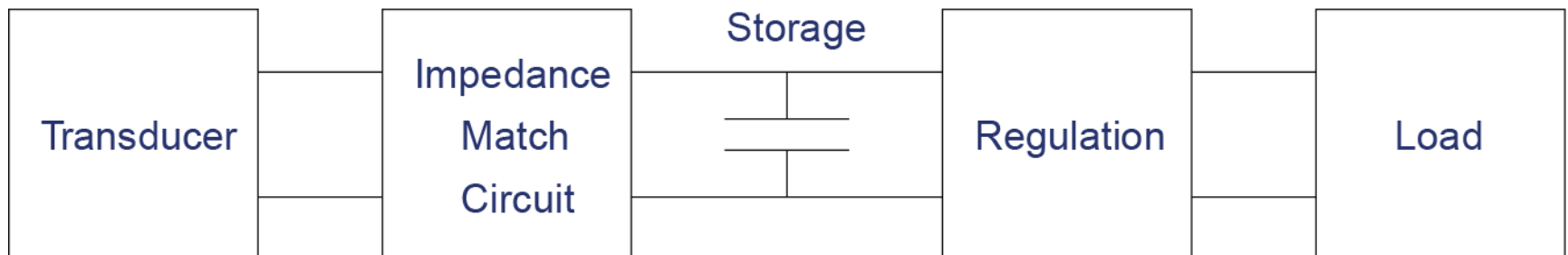
## Transducer Considerations

- With the right electronics, any type of transducer can give any force characteristic
- Easier to make linear damper with electromagnetic and piezoelectric
- Easier to make Coulomb damper with electrostatics
- Scaling laws are important:
  - electrostatic forces dominate at small size
  - electromagnetic are better at larger sizes

*Choice of transducer depends on scale  
and difficulty of implementation*

## Transducer/System Electronics

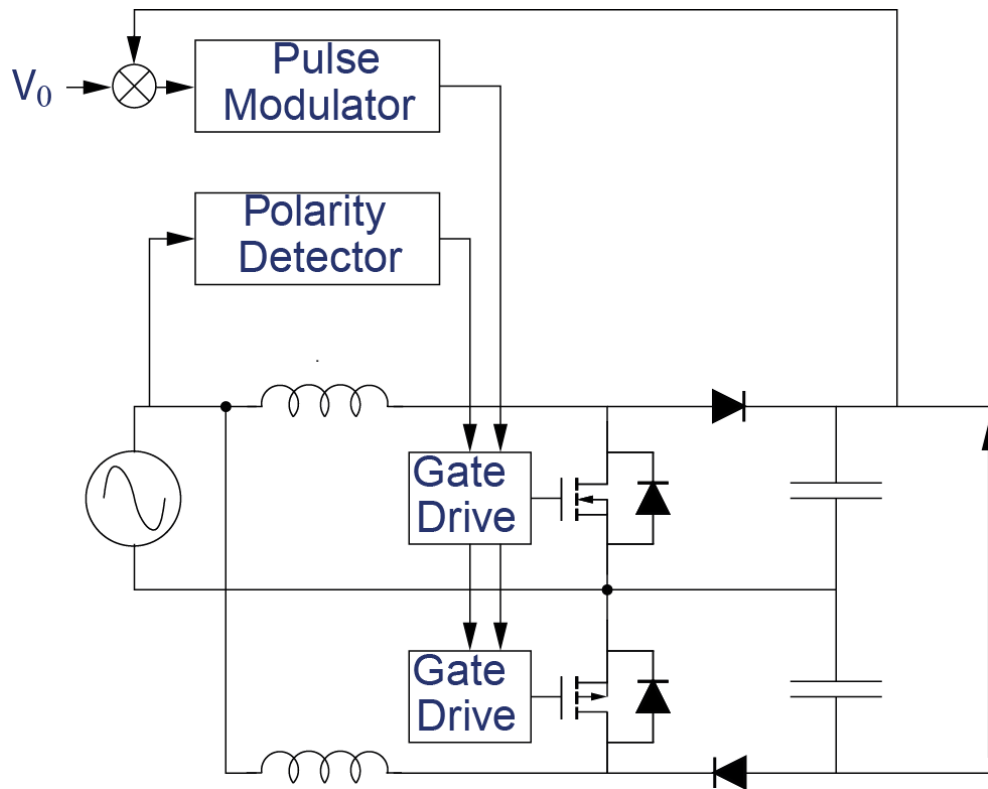
- Need electronics to present optimal load to transducer - impedance match
- This ensures maximum power generation
- Actual load electronics probably not optimal
- Transducer voltage is AC - load wants regulated DC
- Energy storage needed as harvester is intermittent



## Electromagnetic Transducer - requirements

- Voltage from generator is typically small - must be stepped up
- Must draw a specific peak current from generator to keep the optimal damping force
- Resistive is more optimal than an imaginary or non-linear load
- Need power converter with controlled resistive input impedance
- This will create a linear damping

## Proposed El.-mag. Circuit



- Boost topology for step-up
- Draws an in-phase sinusoidal current from a sinusoidal voltage source
- Two converters eliminate need for diode rectifier
- Alter damping by altering duty cycle

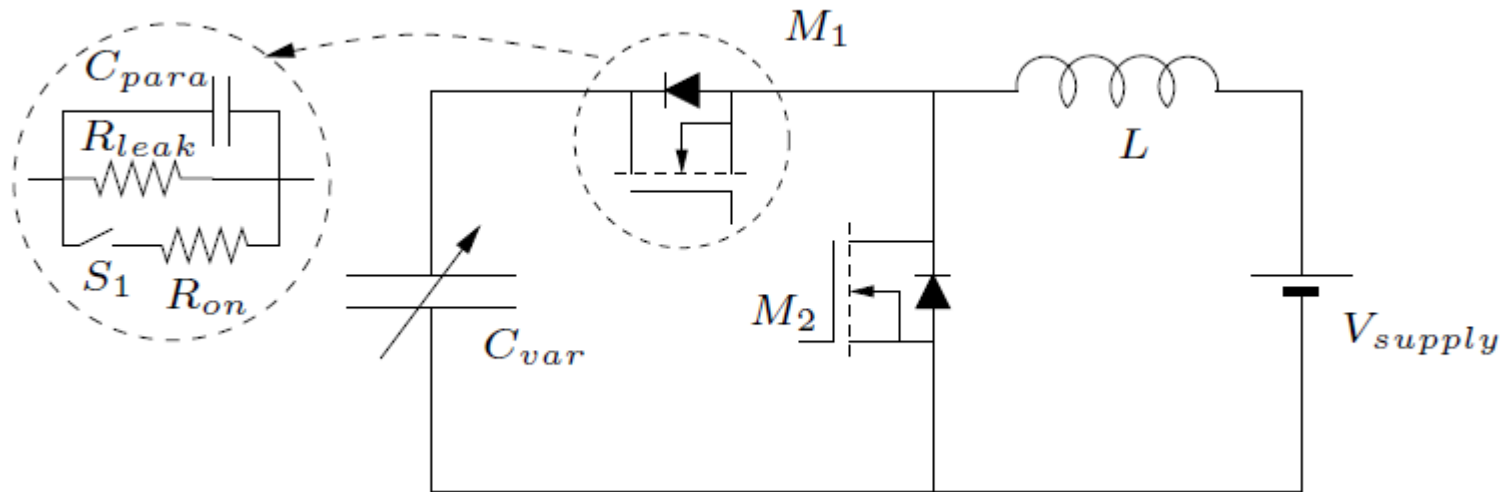
## Summary of Electromagnetic Transducer

- Transducer produces low voltage
- Difficult to rectify with diodes
- Control coil current to control damping force
- Makes a linear transducer (assuming coil inductance ignored)
- Scaling laws suggest it's ok for macro-scale, but not micro-scale

## Electrostatic Transducers - requirements

- Capacitor must be precharged to correct voltage
- Voltages from generator tend to be very high. Step down circuitry required.
- The capacitor must be electrically isolated for const Q part of the cycle -  $10^{12}\Omega$  blocking impedances required.
- Trade-off: Large area semiconductors keep conduction loss low but have high parasitic capacitance and leakage.

## Electrostatic Transducers – proposed circuit



- Buck topology
- Allows controlled charge and discharge of variable C



## Summary of Electrostatic Transducer

- Transducer produces very high voltage
- Difficult to block with integrated pn junctions
- Control capacitor priming voltage to control damping force
- Makes a non-linear transducer (Coulomb-force)
- Scaling laws suggest it's ok for micro-scale, but not macro-scale

## Piezoelectric Transducer

- Typically cantilever structures to give mechanical gain
- Makes it easier to get a large strain on the piezo
- Using a resistive load (resistive input converter) often cannot achieve enough damping
- Especially at high frequency due to shunt  $C_0$
- Voltages and currents are reasonable

*Let's look at a specific example to tie all we have done together*

# Piezoelectric Harvester System Example

*Power-extraction circuits for piezoelectric energy harvesters  
in miniature and low-power applications*

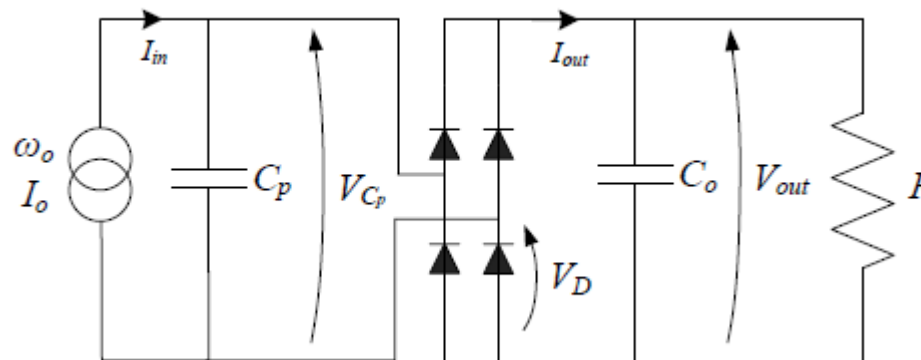
*J Dicken, PD Mitcheson, I Stoianov, EM Yeatman*

*Power Electronics, IEEE Transactions on 27 (11), 4514-4529*

# Piezoelectric Harvesters

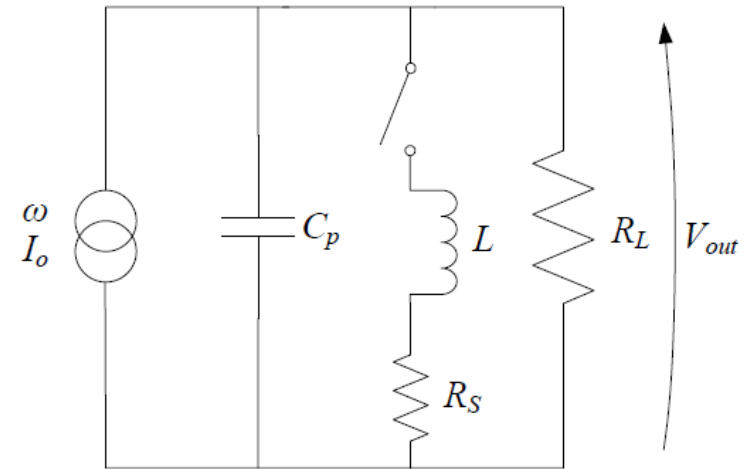
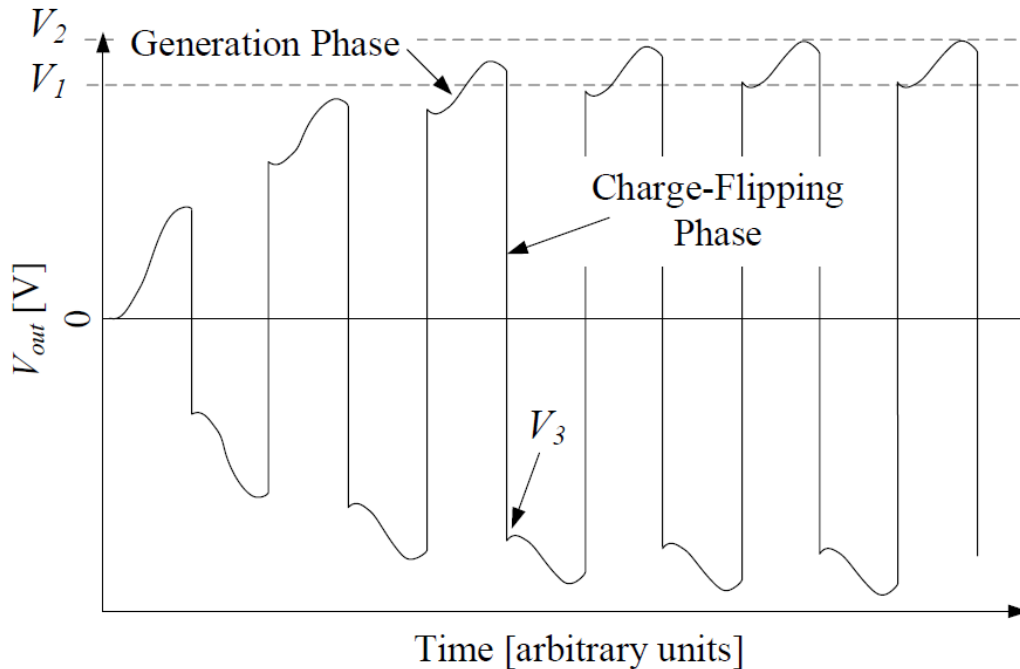
Piezoelectric harvesters produce AC outputs

- Must have rectification
- May require step up or down depending on open circuit voltage of piezo



- But can we extract maximum power?
- As we know, piezo material coupling can be low. This means it is often hard to achieve the optimal electrical damping

## Modify the voltage waveforms on the piezo



Often referred to as SSHI

- Charge on capacitor is resonantly flipped at voltage peaks
- Even in open circuit, a finite maximum amplitude is reached due to finite circuit Q-factor

## It's a Coulomb Damper now!

Remember the force on the piezoelectric material is:

$$F(t) = K_{PE}z(t) + \alpha V(t)$$

Where  $K_{PE}$  is the beam short circuit stiffness.

This can be written as:

$$F(t) = K_{OC}z(t) + \alpha Q(t) / Cp$$

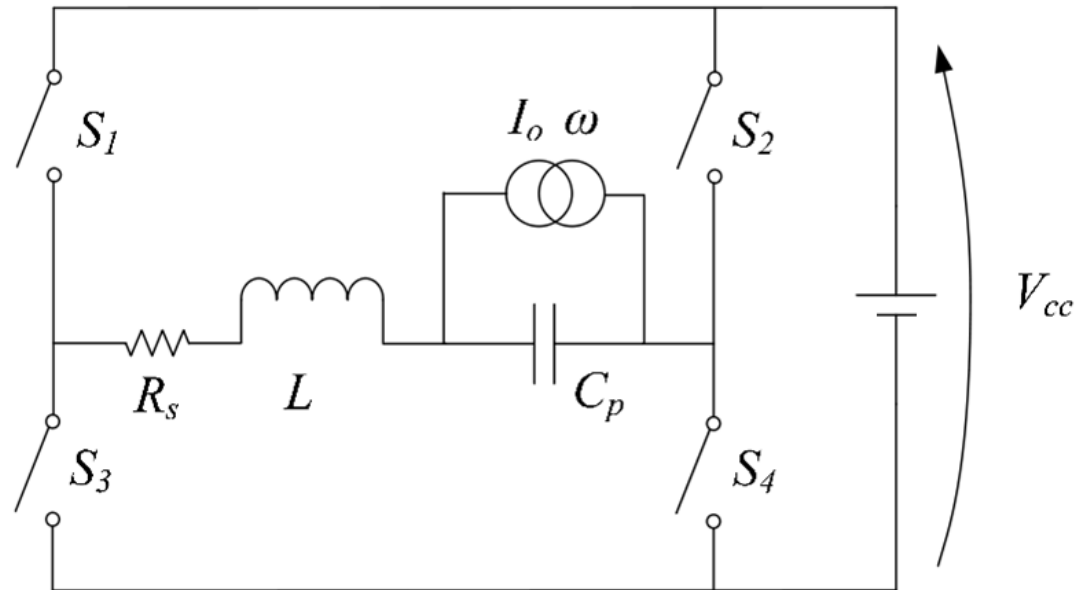
Where  $K_{OC}$  is the open circuit stiffness and  $Q(t)$  is the external charge places on the piezoelectric material

- Hence, pushing a fixed  $Q$  onto the capacitance at the star and end of the cycle can create a Coulomb damper whose value we can control!

## Single Supply Pre-biasing Circuit Overview

Simplified and improved circuit to achieve the waveform modification

- Single source pre-bias circuit
- Source supplies pre-charge
- Generated energy returned to same source
- Can be made diode-less (with no free wheeling currents) if  $V_{CC}$  is optimally set



*Let's see how it works...*

## Power Output Formula

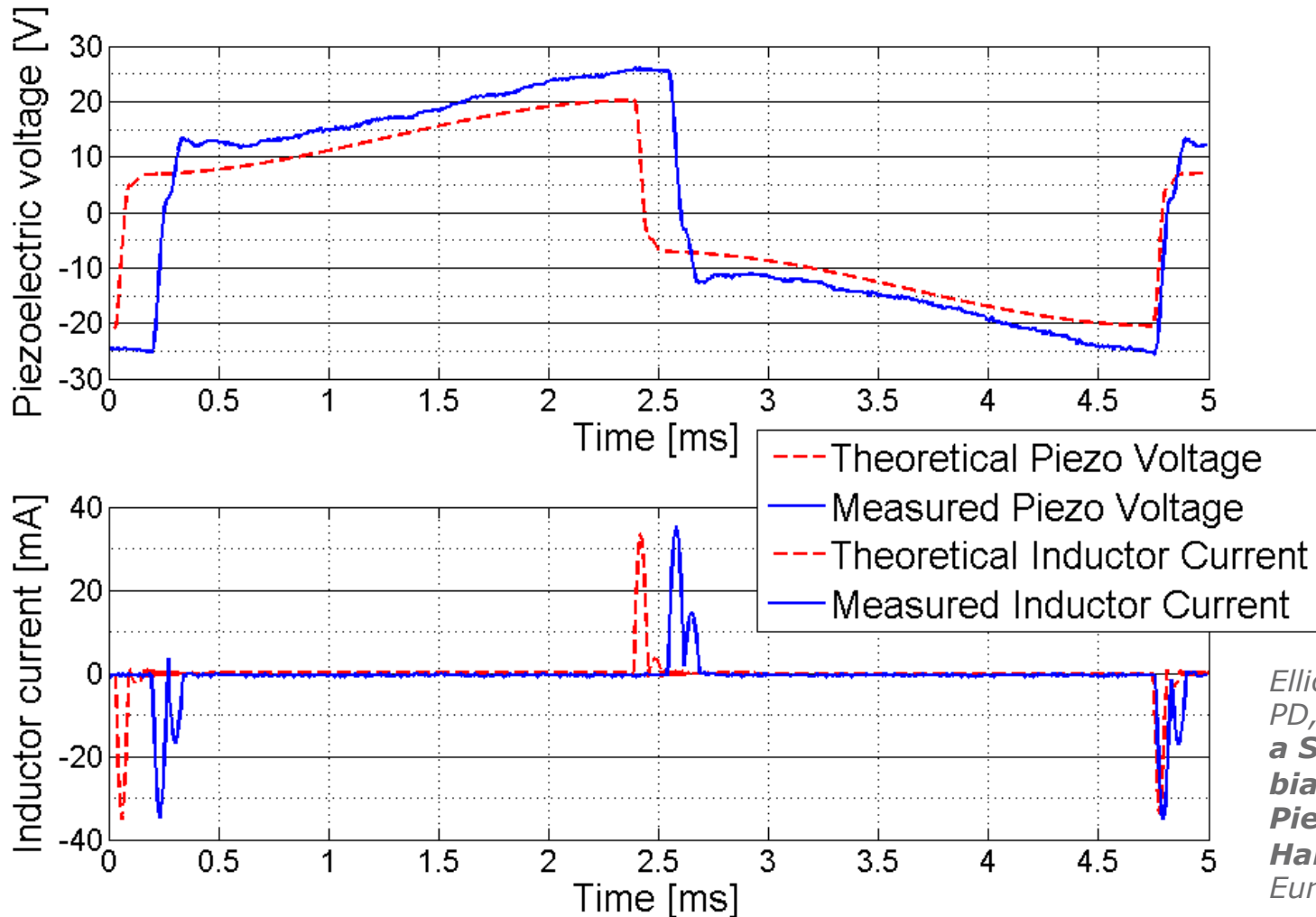
$$P_{\max} = V_{po}^2 f_o C_p \left( \frac{8Q}{\pi} \right)$$

- $V_{po}$  is the open circuit voltage of the piezo
- $f_o$  is the mechanical excitation frequency
- $Q$  is the quality factor of the resonant charging path
- $C_p$  is the capacitance of the piezo

*Dicken J, Mitcheson PD, Stoianov I, et al, **Power-Extraction Circuits for Piezoelectric Energy Harvesters in Miniature and Low-Power Applications**, IEEE Transactions on Power Electronics, 2012, Vol:27, Pages: 4514-4529, ISSN:0885-8993*



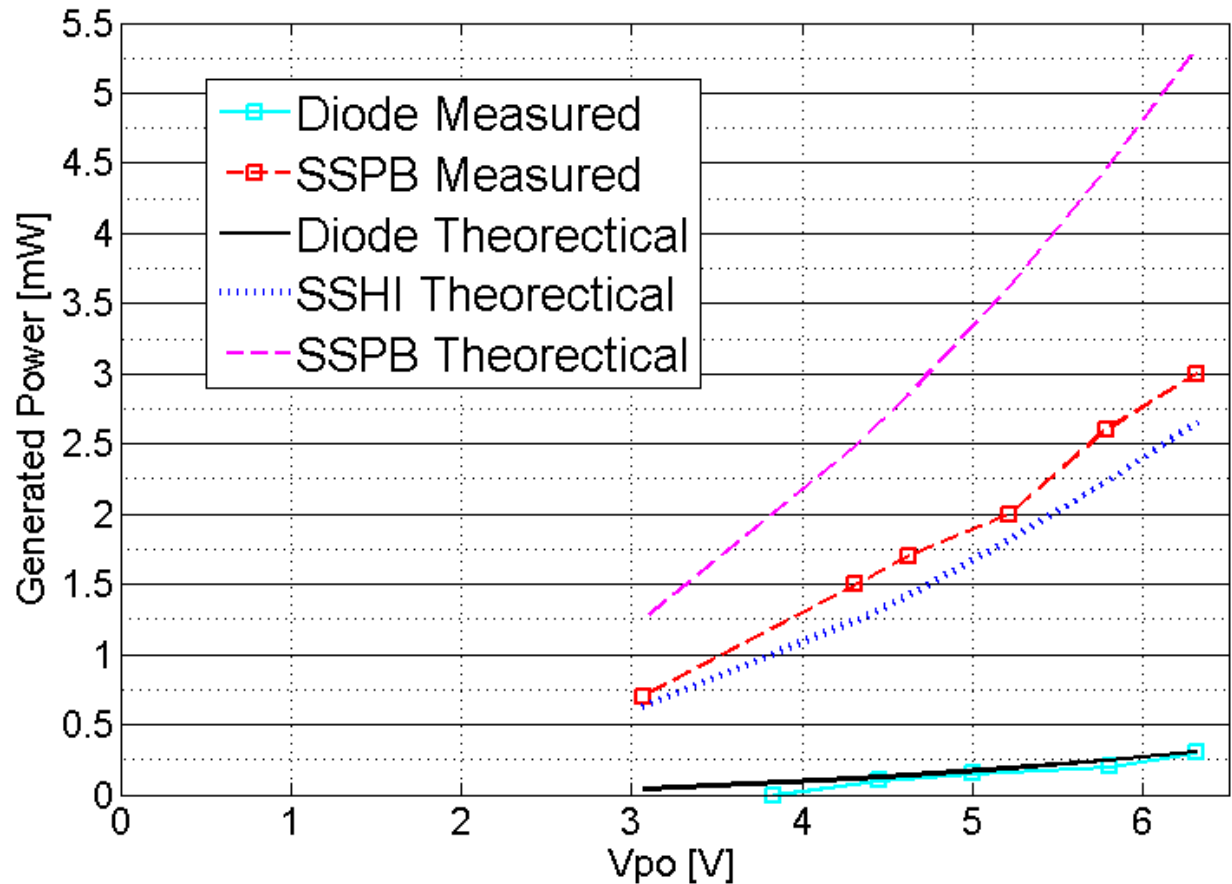
## Results - Waveform



*Elliott ADT, Mitcheson PD, Implementation of a Single Supply Pre-biasing Circuit for Piezoelectric Energy Harvesters, Eurosensors 2012, 2012*

## Results – Technique Comparison

- 400  $\mu$ W control power consumption
- 2.6 mW useful power generated
- 6 times more power extracted than diode rectifier
- Theoretically twice as much power as SSHI



Measured SSPB = 1.13 x SSHI(theoretical)

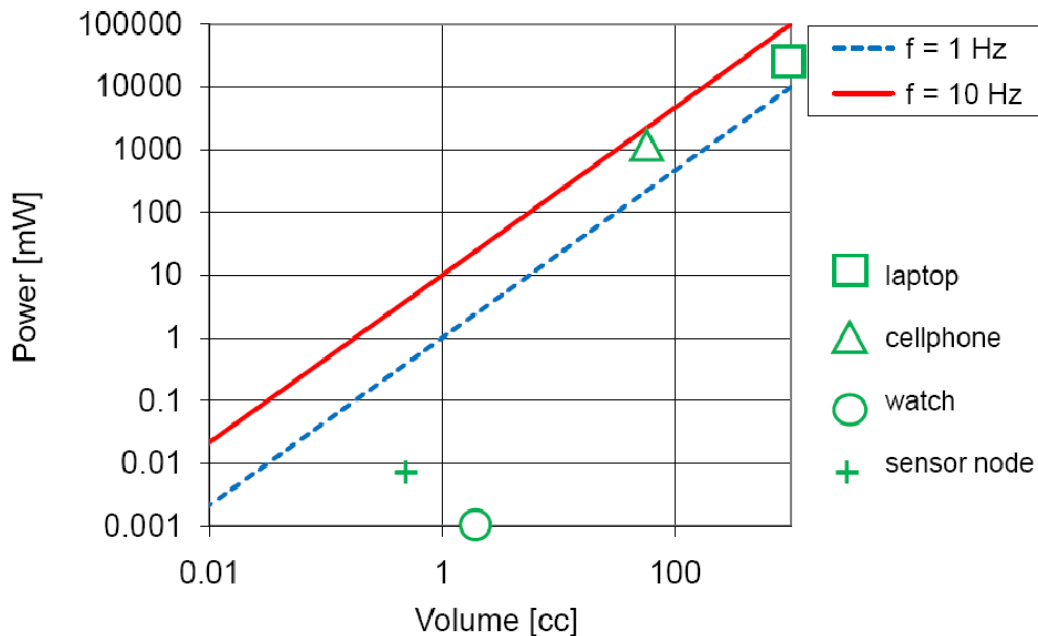
## Conclusions

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- Motion-driven energy harvesters are still performing at a level far below what is theoretically achievable
- Current performance is adequate for some applications such as machine monitoring, and commercial solutions are emerging here
- Significant improvements in performance will be required before harvesting power from human body motion can become viable
- Power conditioning is very important in making a working energy harvester – and a real challenge
- The **system** is important – and being able to analyse the system including the strong link between electric circuit and mechanical systems is important

# Why is it difficult to realise a self powered WSN?

*What can we power from low frequency vibrations?*



*• 1g acceleration*

*• Watch relatively easy to power*

*• Sensor node is around 2 orders of magnitude harder*

*• Forget the laptop and cell phone for several years...*

*We have to optimise the system globally to have a chance of making it functional*

## Acknowledgements

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Eric Yeatman, Andrew Holmes, Tim Green, Tzern Toh, Kondala Rao, Lauriane Thorner, James Dicken, Peng Miao, Bernard Stark, Anisha Mukherjee, Alwyn Elliott

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